1 Original Research Article

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3 APPRAISING OF AIR CONDITIONAL SYSTEM

4 CONDENSATE DISCHARGE RATE IN

5 SOUTHWEST, NIGERIA

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ABSTRACT

Aims: The aim of this research is to ascertain the condensate discharge rate from a split air conditional system with a view to channeling the condensate into a storage tank for other useful purposes.

Study Design: Quantitative study. Relevant data on condensate discharge rate was collected.

Place and Duration of Study: Department of Mechanical Engineering, The Federal University of Technology, Akure, Ondo State, Nigeria, between 16th to the 22nd of November, 2015.

Methodology: The method used consists of data collection and readings such as outdoor dry bulb temperature, dew point temperatures, relative humidity, and condensate volume. It features the calculation of air conditioning load estimation, data analysis, and cost analysis of integrating condensate collection which is essential for the completion of this study.

Results: The study discovered that about three thousand four hundred and eight-six (3,486) litres of condensate are being wasted on a weekly basis within the School of Engineering and Engineering Technology, The Federal University of Technology, Akure. This figure indicates the amount of reclaimed water source that is not in use.

Conclusion: The study has shown that the imbalance in water supply and demand currently experienced at School of Engineering and Engineering Technology building can be managed effectively if condensate water is properly collected. The condensate collected should be channelled by gravity flow through 32mm PVC pipes into a 5,000 Litres storage tank within the building and a Duty/Assist 2Hp Booster pump is to be installed to transfer the condensate into the existing plumbing works to the restrooms and laboratory for good use.

Keywords: Air-conditional System, Cooling Load Estimation, Reclaim Water, Condensate Discharge.

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15 **1. INTRODUCTION**

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17 Air conditioning can be referred as the process of simultaneous control of temperature, 18 humidity, cleanliness and air motion for the comfort of human beings, animals or for the proper attainment of some industrial or scientific process [1]. The by-product of the 19 20 simultaneous cooling in the air-conditioning system is the condensed water which is formed 21 from moisture in the air. Therefore, condensate from air conditional is classified under the 22 definition of alternative on-site sources of water such as rainwater to bolster water and 23 decrease energy dissipated by town water operations [2]. The most efficient and vital use of 24 condensate harvested from commercial or industrial buildings includes; toilet flushing, 25 irrigation, ornamental water features, and process water, such as that used in manufacturing 26 and makeup water in cooling towers.

27 It was reported by [3] that high temperature and humidity during summer months contribute 28 greatly to condensate production amounting to 6 to 7 ml/s/1000 m² of the cooled area of 29 buildings. The weather in Ondo State, Southwest, Nigeria is characterised by maximum daytime temperatures which rarely exceed 34 °c and low as 22 °c, a mean annual relative 30 humidity of about 80 % [4]. Therefore, this makes it a good fit for condensate water recovery. 31 32 Unfortunately, most buildings in Nigeria allows the condensate to typically drained away to an open ground as waste, which is an abundant source of reclaimed water. Hence, this 33 34 paper aims to appraise the condensed water collected from a split-type air-conditioning 35 system in a Nigerian office space.

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37 2. LITERATURE REVIEW

38 Many types of research have been identified to determine the advantages of condensate 39 recovery system in different places of the world. This study investigates some of this source 40 of these condensate recovery system and the importance. [5] have found a modelling 41 approach for determining condensate collection for an Institutional Building in Doha, Qatar. 42 Results indicated that as the dew point temperature increased, the corresponding 43 condensate of 660,000 gallons of water was captured from the condensate drains of the six 44 hundred installed tons of air conditioning on a large commercial building in a location with 45 one fourty days of dew point temperatures above 60°F. [6] reiterated the means of using the air-conditioning condensate as an added source of water. The results revealed that the 46 enormous cost of implementing a condensate collecting system is still acceptable for 47 48 buildings with high fresh air percentage. It was further observed that weather condition and space occupancy impact the volume of condensate generated. [7] presented a study of 49 50 condensate modelling system for high performing buildings. The objective of the work was to 51 develop a modelling system to estimate the amount of condensate from an air-conditional 52 unit on an existing building. Some of the readings collected in the research work include; 53 outdoor temperature, dew point and relative humidity of the fresh air and inside air 54 conditional unit. The result showed that the total amount of condensate collected was 55 171,793 gallons during the summer season with 3.5 gallons per hour. [8] founded a 56 modelling prediction technique for an outdoor air-conditional unit with energy recovery, while 57 using annual daily readings of average temperature and humidity data. The analysis of the result indicated that the condensate production from the case study building's large 58 59 dedicated outdoor air-conditional unit can completely supplement the annual daily water closet and urinal water demand with 6.12×10^6 L of excess, which could be used to 60 supplement landscape irrigation system. [9] research study revealed that the design of 61 62 HVAC systems in a sustainable laboratory facility will assist in high amounts of condensate recovery. The author revealed that utilizing weather data information in a newly built 63 64 laboratory in Atlanta, and an estimated cooling load, the appraised condensate collection 65 from the cooling coils exceeded 3028330 L every year.

67 3. METHODOLOGY

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This study approached the subject of condensate collection with three specific purposes. It started with the development of a technique to identify the total volume of condensate collected using correlations with weather data. The next stage involves evaluating the method associated with a typical condensate collection system. Finally, the result collected was analysed for generalizations regarding environments where condensate collection would be recommended for the economic and environmental impact perspective.

76 3.1 Materials

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A split type air-conditional unit (2500 W), graduated measuring cylinder (500 ml), thermohygrometer, collector (25 litres) and drainpipe (using 25 mm PVC adaptor)

3.2 Selection of environment and condensate measurement

82 The study was carried out in an office space within the School of Engineering and 83 Engineering Technology, Federal University of Technology, Akure, Ondo State, Nigeria. The 84 hourly values of dry bulb temperatures and relative humidity with operating conditions of the air handling unit give the typical condensate rate for each hour. The condensate produced 85 86 for each specific hour was added to compute the condensate for that specific day. 87 Precautions taken to enhance accuracy in measurement of condensate includes, calibration 88 of the thermo-hygrometer apparatus used for the study to avoid errors associated with dry 89 bulb temperature and relative humidity readings. The condensate collector was constantly 90 checked to verify that there are no leaks. Although this study only looked at an office space within the school of engineering building, developing correlations between the amount of 91 condensate and weather data parameters allows the results and conclusions to be 92 93 applicable anywhere. The first step toward achieving this goal was to check if the correlation 94 can be derived for the amount of condensate collection with respect to a readily available weather data parameter. The office space can accommodate at least four people and its 95 96 volume was 33.8 m³. Figure 1 shows the room plan of the office space.



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116 **Fig 1. Room plan of the office space**

117 **3.3 Sizing and Installation of the Split Air-Conditioning System**

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119 According to [10], the occupancy activity impacts the fresh air ratio which may differ from 100% fresh air in operations like a kitchen, laundry room, and electrical room (where there is 120 121 a presence of a pure sensible load). To calculate the cooling load of a building's volume, the 122 effect of heat stored in the mass of the building has made it difficult due to its thermal inertia 123 to heat response. Therefore, the sizing of the air conditional system for room conditioning should start with the theory of air conditioning load estimation in buildings and analysis of 124 125 heat transfer in a typical sample of building construction in Akure. In order to achieve this, the Cooling Load temperature difference (CLTD) and Glass Load Factor (GLF) approach 126 127 were developed by ASHRAE. Therefore, the conductive transfer across the wall surfaces or 128 roof is defined in the form;

129 130 131	•	Roof Construction Conventional roof-attic-ceiling combination U = 0.28 W/(m ² K)	Þ
132 133 134 135	•	Wall Construction Brick, insulation, brick wall U = 0.34 W/(m ² K) Partition wall U = 0. 4 W/(m ² K)	
136 137 138	•	Doors Wood, solid core U = 1.82 W/(m2·K)	
139 140 141 142 143	•	WindowsClear louver-pane glass in metal frames3mm thick. $U = 2.84 W/(m2 \cdot K)$ The window glass has a 600 mm overhang at the top.Assume closed, medium-color Venetian blinds.	
145 146 147 148	•	Outdoor Design Conditions A temperature of 33°C dry bulb with a 13 K daily range Relative humidity ratio of 0.0136 kg vapour/kg dry air (23.7 °C wet bulb)	
149 150 151 152	•	Indoor Design conditions A temperature of 22 °C dry bulb Relative humidity ratio of 55%	
153 154 155	•	Occupancy Four persons	
156 157 158 159	•	Appliances and lights 110 W for the light fittings and 150% for the appliances in the office From the sensible, latent and total cooling load!	
160 161 162 163 164 165 166	•	Heat gains For walls, roof, and doors Q = U. A. (CLTD) (1) Where, $U = coefficient of heat transfer W/(m^2k); A = area of surface, m^2,CLTD = Cooling Load Temperature Difference, K$	

167 168 169 170	•	For Walls Q = 0.34×33.8×14 Q = 161 W	
171 172 173 174	•	For Partition Q = U. A. (CLTD) $Q = 0.4 \times 33.8 \times 4$ Q = 54 W	
175 176 177 178 179 180	•	For Roof Q = U. A. (CLTD) Q = 0.28×10.86×27 Q = 82 W	D
181 182 183 184 185	•	For Doors (Two numbers) Q = 2 (U. A. (CLTD)) Q = 2 (1.82×1.17×14) Q = 60 W	
186 187 188 189 190 191	•	For Window – considering the overhang louver and curtain (H=2m) Q = 2(A. (GLF)) Where GLF = glass load factor; A = area of surface, m^2 , Q = 2 (1.416 × 141) Q = 399 W	(2)
192 193 194 195 196 197	•	For Occupancy Q = 67 Wn Wn = no of persons $Q = 67 \times 4$ Q = 268 W	(3)
198 199 200 201 202 203 204	•	Volumetric airflow rate for summer sensible heat $\dot{V} = ACH. V. 1000/3600$ $\dot{V} = Volumetric airflow rate I/s; V = Volume of room m3$ ACH = Summer air change rate I/h $V = 0.5 \times 33.8 \times 1000/3600$ V = 4.69 I/s	(4)
205 206 207 208 209 210 211		Infiltration $Q = 1.2$, \dot{V} . Δt Where Δt = Outside and inside design temperature respectively (°C) Where \dot{V} = Volumetric airflow rate I/s $Q = 1.2 \times 4.69 \times (33-22)$ Q = 62 W	(5)
212 213 214 215 216 217 218 219	•	Total Heat Gain Q = Latent heat + sensible heat Qtotal= 1303W	(6)

220 **3.4 Cost Estimate**

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This is the overview of the cost of materials needed for installing piping fittings, accessories and electrical connectors in condensate collection. The estimated cost of integrating condensate collection to existing plumbing works of toilet and laboratory in the engineering building of the Federal University of Technology, Akure is shown in Table 1.

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227	Table	1.	Cost	analysis	of	integrating	condensate	collection	to	existing
228		e	engine	ering build	gnit	1				

S/N	Materials	Specifications	Quantity	Unit Cost	Total Cost
	Description			(N)	(N)
1	GPee Storage Tank	5000 L	1	70,000.00	70,000.00
	with all necessary				
	accessories				
2	Lowers Posster	2 Hp. 20 I /min	1 oot	210,000,00	210 000 00
2		2 Hp, 20 L/IIIII	I SEL	210,000.00	210,000.00
	vvater Pump		$\mathbf{\Omega}$		
	(Duty/Assist)				
3	Electrical Installation	lot	2		75,000.00
	and control panel				
_		\sim			
4	Plumbing Work and	32 mm PVC			50,000.00
	Piping Fittings	X .			
5	Contingencies				100 000 00
5	Contingencies				100,000.00
	TOTAL				N 505,000.00

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230 4. Results and Discussion

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This project collected condensate data for a period of 1 week in November 2015. After the 232 problem with continuous power failure was corrected, data collection really started on 233 November 16th, 2015. The results presented here cover a week period from the 16th to the 234 22nd of November, 2015. The concurring outdoor dry bulb temperature, relative humidity, and 235 dew point temperatures during this period were also recorded. During this period, 236 237 condensate collection varied from almost no quantity to what appears to be highest for the 238 100% outdoor air unit. The split air conditioning unit serves only the office under study and it is a one fan unit with provisions for outside air, return air and nominal cooling capacity of 239 240 2500 W. The average outdoor temperature and humidity during this period ranges from 52 $^{\circ}$ C – 68 % and 26 $^{\circ}$ C – 31 %. The result in Table 2 showed that a total of 42.03 Litres of 241 242 condensate was harvestable between the time periods from the split air conditioning unit 243 serving the office under study. The result collated also indicated that the figure could rise to 244 as much as 3,486 L a week if the entire 83 numbers of split air conditioning unit serving laboratory and offices within the building are functioning. This figure indicates the amount of 245 reusable water source that is not in use. The total volume of condensate harvested within 246 247 the school of engineering and engineering technology building will be vital in complementing 248 the primary water source such as watering of the landscape area, the source of water in the 249 toilets and rinsing laboratory apparatus. The summary of the condensate collected for seven 250 days is shown in Table 1.

DAY	TIME		Average Relative Humidity	Average Temperature (°C)	Average Dew Point	Condensate Collected (Litres)	
	From	То	- (70)		(0)		
16/11/2015	8:00am	4:00pm	57%	26	23	6.17	
17/11/2015	8:00am	4:00pm	68%	29	21	6.85	
18/11/2015	8:00am	4:00pm	52%	29	21	4.74	
19/11/2015	8:00am	4:00pm	59%	30	22	5.83	
20/11/2015	8:00am	4:00pm	64%	29	21	5.91	
21/11/2015	8:00am	4:00pm	65%	29	21	6.25	
22/11/2015	8:00am	4:00pm	60%	31	23	6.28	
			\mathcal{A}	Total Condensa Collected in SEE of A/C Unit)	te ET (x 1 nos	42.03 L	

Net Total for 83 no's of A/C

unit Installed & Functioning

3.486 L

Table 2. Showing the evaluation of condensate collection for five days

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253 It was also observed in Table 2 that as the average relative humidity increased, there was a 254 corresponding increase in the amount of condensate collected. Thus, the greater the relative 255 humidity at a given temperature, the greater volume of the condensate formed and vice 256 versa. The result agrees with the works of Lawrence et al. (2009) and Johnson (2008) who found that high relative humidity with corresponding dry bulb outdoor temperature and dew 257 258 point gives rise to high volume of condensate that will be collected from an air conditioning 259 system. Finally, as the temperature rises during the daytime (26 °C - 31 °C), the relative humidity for the corresponding time decreases (72 % - 53 %) and It can be said that the 260 261 temperature is generally inversely proportional to the relative humidity.

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263 **5. Conclusion**

The aim of the study is to apply recent techniques in appraising condensate collection potential and to determine if condensate collection would be recommended for the main water source. The application of reclaimed water source such as collected condensate is one approach for reducing long-term potable water consumption. The imbalance in the water supply and demand currently experienced at SEET building can be managed effectively if 269 the condensate water is properly collected. Water and energy are conjoint and hence it is 270 essential that building operators and industry leaders understand that water conservation is 271 as important as energy conservation. The study discovered that about three thousand four 272 hundred and eighty-six (3,486) litres of condensate are being wasted on a weekly basis 273 within the School of Engineering and Engineering Technology, FUTA. Therefore, the 274 condensate collected should be channelled by gravity flow through 32mm PVC pipes into a 275 5,000 Liters storage tank within the SEET building and a Duty/Assist 2Hp Booster pump is to 276 be installed to transfer the condensate into the existing plumbing works to the restrooms and 277 laboratory for good use.

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