1	Original Research Article
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3	Heat Transfer of Mixed Convection Electroconductivity Flow of Copper
4	Nanofluid with Different Shapes in a Porous Micro Channel Provoked
5	by Radiation and First Order Chemical Reaction
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8	Abstract:
9	This work presents a theoretical description of different shapes of copper
10	nanoparticle in water based fluid. Analytical solution of the governing
11	hydrodynamic equations and graphs plotted using Mathematica 9.0 software,
12	showed that heat transfer is rapid due to the presence of radiation. Increase
13	in radiation and chemical reaction also led to a corresponding increase in the
14	temperature and concentration profiles of the nanofuid respectively. For the
15	velocity profile of the nanofluid, nanoparticles volume fraction is the only
16	parameter that its increase, increases the velocity profile of the copper
17	nanofluid but Reynolds number, Grashof's number and electroconductivity,
18	result to decrease in the velocity profile of the copper nanofluid. The effect
19	of Nusselt number, Sherwood number and skin friction on the nanofluid is
20	also determined.
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22	Key Words-: Heat transfer, Copper, Nanofluid, Porous Channel, Water Based
23	Fluid
2425	Notation
26	ρ_{nf} density of nanofluid
27	P pressure of fluid
28	u dimensionless fluid velocity
29	μ_{nf} dynamic viscosity of nanofluid
30	$(\rho\beta)_{nf}$ thermal expansion coefficient of nanofluids due to temperature
31	$\sigma_{\scriptscriptstyle \infty}$ constant fluid electronconductivity
32	U plate velocity
33	k_0 porosity of the medium
34	k_{nf} thermal conductivity of nanofluid
35	q radiative heat f lux
36	g acceleration due to gravity
37	k_f thermal conductivity of base fluid

 ϕ

 $ho_{\scriptscriptstyle f}$

nanoparticles volume fraction

density of base fluid

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density of nanoparticles
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                 heat capacitance of nanofluids
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                 volumetric coefficient of nanoparticles
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                  volumetric coefficient of base fluid
44
                 specific heat capacity of base fluid
      (C_p)_s
45
                specific heat capacity of nanoparticles
      C'
46
                concentration of fluid
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      C
                dimensionless concentration of fluid
48
      Τ
                temperature of fluid
49
      \theta
                dimensionless temperature of fluid
50
      Re
                Reynolds number
                 Peclet number
51
      Pe
52
      Gr_{T}
                 thermal Grashof number
53
                 concentration Grashof number
      Gr_{C}
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      N
                dimensionless radiation term
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                dimensionless chemical reaction term
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                 chemical reaction term
57
                 thermal conductivity ratio
58
                 molecular diffusivity
       D_{nf}
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                 fluid velocity
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1. Introduction

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Thermal conductivity of heat transfer in fluids is key in the development of energy efficient heat transfer equipment. Conventional heat transfer fluids such as water, oil, ethylene glycol mixtures to mention few, are poor heat transfer fluids. With global need for improvement, to develop advanced heat transfer fluids with significantly higher thermal conductivities than those presently available is of utmost necessity. Touloukian et al (1970) has opined that at room temperature, metals in solid form have orders of magnitude higher that those of solids. The new class of heat transfer fluids that are engineered by suspending nanometersized particles in conventional heat transfer fluids whose averaged sized particles is below 50nm is termed nanofluids Choi (1995) The reality is that in today's science and technology, size does matter, therefore modern fabrication technology provides great opportunity to actively process materials at the micro and nanometer scales. The impact of this new heat transfer technology is important due to its performance in thermal conductivity and viscosity which led to improved heat transfer and stability, reduced pumping power, minimal clogging and miniaturized systems as well as cost and energy savings (Choi et al 1992a and 1992b). The different composition of nanoparticles in base fluids to form nanofluids has its application in power generation and electronic equipment just to mention few. The applications of nanofluids is predicated on the desirable propreties or qualities following Mukherjee and Paria (2013) as follows (i) rapid

increase in thermal conductivity (ii) ultrafast heat transfer ability (iii) reduce pumping power (iv) reduce friction coefficient (v) reduce clogging in microchanels (vi) improved stability than other colloids and (vii) improved lubrication. Feng et al (2006), examined the preparation of gold, silver and platinum nanofluids using aqueous organic phase transfer method. Yu et al (2010) Wei et al (2010) and Zhu et al (2007), prepared Copper oxide nanofluid and use amoniun citrate to prevent the growth and aggregation of nanoparticles, resulting in a stable CuO aqueous nanofluid with higher thermal conductivity. Other works such as (Hwang, et al (2007) and Li et al(2007), used spectral analysis method to detect stability of nanofluids. Experimental and theoretical studies are abounded on conductivity, viscosity and stability of nanofluids and its aggregate nanoparticles. Duncan and Rouvray (1989), Gliter(1989) and Hashin and Shtrikman (1962) examined nanofluid materials and made far reaching deductions from their findings. The model proposed by Hamilton and Crosser (1962) to predict the thermal conductivity of nanofluid, containing large agglomerated particles and that obtained from experimental results were compared and it fits well, but divergence is observed at low volume fractions. The implication of this observation is that particle size and shape dominate thermal conductivity of nanofluids (Li 1998). Studies on nanoparticles of spherical shapes are many but limited in applications and significance, Aaiza et al (2105). As a result of this assertion, a non spherical shaped nanoparticle is chosen for this study in four different shapes, namely, platelet, cylinder, blade and brick. The choice of non spherical shape nanoparticle is predicated on the work of Aaiza et al (2015). where they mentioned desirable properties in cancer treatment. Asma et al (2015), studied free convection flow of nanoparticles including ramped wall temperature using five different types of spherical shape nanoparticles and reported that the solution of the governing equations was exact. Heat transfer due to mixed convection is experienced in many physical situations and occur as the flow in a channel due to the process of heating or cooling of the channel walls. This experience is a combination of free convection and forced convection. Some researches have been reported, they include (Sebdani et al (2012), Sheikhzabeh et al (2012), Nadeem and Saleem (2014) and Al-Salem et al (2012) in which mixed convection and nanofluid were discussed in different nanoparticles shape and configurations. Aaiza et al (2105), investigated water based nanofluid and ethylene glycol based nanofluid and reported that viscosity and thermal conductivity are the most prominent parameters responsible for different results of velocity and temperature. The present study is to examine the effect of first order chemical reaction and electroconductivity on radiative heat transfer in mixed convection flow in a micro porous channel with different shapes of copper (Cu) in water based nanofluid. The Cu in water based nanofluid preparation and sphericity is reported in Yimin and Li (2000). The reason is that Cu as a nanoparticle, possesses higher thermal conductivity and stability than other nanofluids. The focus is to consider non spherical shaped nanofluid under the no slip boundary conditions with bounding walls of the channel at rest, the upper wall in motion and lower at rest and both walls are in motion. Solutions to velocity, temperature and concentration profile with graphical results and

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parameters of interest discussed, which is an extension of the work of Aaiza et al (2105)

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2. Formulation of the problem

- The assumption of the effect of induced magnetic field is small, therefore, neglected.
- Also the usual Boussinesq approximation is assumed. The no slip condition at the
- boundary wall is considered. The x-axis is taken along the flow and y-axis is taken
- normal to the flow direction. The governing hydrodynamic equations are given as

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$$\rho_{nf} \frac{\partial u'}{\partial t'} = -\frac{\partial p}{\partial x'} + \mu_{nf} \frac{\partial^2 u'}{\partial y^2} - u \left(\frac{\sigma_{\infty}}{U} + \frac{\mu_{nf}}{k_0} \right) + (\rho \beta_T)_{nf} g(T - T_0) + (\rho \beta_C)_{nf} g(C - C_0)$$
(1)

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$$\left(\rho C_p\right)_{nf} \frac{\partial T}{\partial t'} = k_{nf} \frac{\partial^2 T}{\partial v^2} - \frac{\partial q}{\partial v}$$
 (2)

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$$\left(\rho C_{p}\right)_{nf} \frac{\partial C'}{\partial t'} = D_{nf} \frac{\partial^{2} C'}{\partial y^{2}} - k_{r}^{2} C'$$
 (3)

With the boundary conditions

143
$$u'(0,t) = 0, \quad u'(d,t) = 0,$$
 (4)

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$$T(0,t) = T_0$$
, $T(d,t) = T_w$ (5)

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$$C'(0,t) = C_0$$
, $C'(d,t) = C_w$ (6)

- where following Boricic et al. (2005), the fluid electroconductivity is assumed to be of
- the form $\sigma_{\infty} \left(1 \frac{u'}{U} \right)$ but for physical exigency and mathematical amenability, it is
- approximated to the form in equation (1) Ngiangia and Harry (2017).

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- According to a model proposed by Hamilton and Crosser (1962), the thermal
- 153 conductivity and dynamic viscosity is assumed valid for both spherical and non spherical
- shapes nanoparticles. The model is stated as

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$$\mu_{nf} = \mu_f \left(1 + a\phi + b\phi^2 \right)$$
 (7)

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$$\frac{k_{nf}}{k_f} = \frac{k_s + (n-1)k_f + (n-1)(k_s - k_f)\phi}{k_s + (n-1)k_f - (k_s - k_f)\phi}$$
 (8)

- where $n = \frac{3}{\psi}$ is the empirical shape factor and ψ is the sphericity, a ratio of surface area
- of sphere to surface area of real particle with equal volumes as in **table 1** with **a** and **b** as constant empirical shape factors.

Another expression by Wasp(1977) to determine the effective thermal conductivity of

solid-liquid mixture is given as

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$$\frac{k_{eff}}{k_f} = \frac{k_s + 2k_f - 2\phi(k_f - k_s)}{k_s + 2k_f - \phi(k_f - k_s)}$$
 (9)

This is a special case of equation (8) with sphericity 1.0

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From equations (1), (2) and (3), ρ_{nf} , $(\rho\beta)_{nf}$ and $(\rho C_p)_{nf}$ following Asma et al (2015) is

derived as

$$\rho_{nf} = (1 - \phi)_{\rho_f} + \phi \rho_s$$

172
$$(\rho\beta)_{nf} = (1-\phi)(\rho\beta)_f + \phi(\rho\beta)_s$$

$$(\rho C_p)_{nf} = (1-\phi)(\rho C_p)_f + \phi(\rho C_p)_s$$
(10)

In the work of Makinde and Mhone (2005), the plates temperature T_0 and T_w are usually

high and produces radiative heat transfer. According to Cogley et al (1968), for optically

thin medium with relatively low density, the radiative heat flux is given by

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$$177 \qquad \frac{\partial q}{\partial y} = 4\delta^2 \left(T - T_0 \right) \tag{11}$$

where δ is the radiation absorption coefficient

179 Substituting equation (11) into equation (2) and using the following dimensionless

180 variables

$$x = \frac{x'}{d}, y = \frac{y'}{d}, u = \frac{u'}{U}, t = \frac{t'U}{d}, \frac{\partial p}{\partial x'} = \lambda \exp(i\omega t), \theta = \frac{T - T_0}{T_w - T_0}, \text{Re} = \frac{Ud}{\mu_f}$$

181
$$k = \frac{k_0}{d^2}, Gr_T = \frac{g\beta_f d^2 (T_w - T_0)}{\mu_f U}, Gc_T = \frac{g\beta_c d^2 (C' - C_0)}{\mu_f U}, \sigma_0 = \frac{\sigma_\infty \mu_f dt}{U}$$

$$N = \frac{4\delta^2 d^2}{k_f}, Pe_T = \frac{Ud(\rho C_p)_f}{k_f}, Pe_C = \frac{Ud(\rho C_p)_f}{D_{nf}}, k_\infty = \frac{k_r^2 T_0}{\sigma_\infty U^2}, \lambda_n = \frac{k_{nf}}{k_f}$$

182 Further, we define

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$$w_1 = (1 - \phi) + \phi \frac{\rho_s}{\rho_f}$$
 $w_2 = (1 + a\phi + b\phi^2)$ $w_3 = (1 - \phi)\rho_f + \phi \frac{(\rho\beta)_s}{\beta_f}$

185
$$w_4 = \left[(1 - \phi) + \phi \frac{(\rho C_p)_s}{(\rho C_p)_s} \right]$$
 $a_1 = w_1 \text{ Re}$ $a_2 = \sigma_0 + \frac{w_2}{k}$ $a_3 = w_3 Gr_T$

186
$$a_4 = w_3 Gr_c, c_1 = \frac{Pe_c w_4}{\lambda_n} b_1 = \frac{Pe_T w_4}{\lambda_n} b_2 = \frac{N}{\lambda_n}, c_2 = \frac{k_\infty}{\lambda_n},$$

then equations (1), (2) and (3) takes the form

188
$$a_1 \frac{\partial u}{\partial t} = \lambda \varepsilon \exp(i\omega t) + w_2 \frac{\partial^2 u}{\partial y^2} - a_2 u + a_3 \theta + a_4 C$$
 (12)

189
$$b_1 \frac{\partial \theta}{\partial t} = \frac{\partial^2 \theta}{\partial y^2} - b_2 \theta \tag{13}$$

$$190 c_1 \frac{\partial C}{\partial t} = \frac{\partial^2 C}{\partial y^2} - c_2 C (14)$$

191 Subject to the boundary conditions

192
$$u(0,t) = 0, u(1,t) = 1 t > 0$$
 (15)

193
$$\theta(0,t) = 0, \quad \theta(1,t) = 1 \qquad t > 0$$
 (16)

194
$$C(0,t) = 0$$
, $C(1,t) = 1$ $t > 0$ (17)

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3. Method of Solution

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To solve equations (12-14), we assume solution of the form

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200
$$u(y,t) = [u_0(y) + \varepsilon \exp(i\omega t)u_1(y)]$$
 (18)

201
$$\theta(y,t) = [\theta_0(y) + \varepsilon \exp(i\omega t)\theta_1(y)]$$
 (19)

202
$$C(y,t) = [C_0(y) + \varepsilon \exp(i\omega t)C_1(y)]$$
 (20)

203 where ε is a small parameter

204

- Case 1: Both walls of the channel are kept stationary, while the temperature of the upper wall of the channel is assumed constant, the lower wall has uniform temperature and the boundary conditions conform to equations (18-20).
- We substitute equation (20) into equation (14) and simplify, the result is

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$$210 C_0''(y) - c_2 C_0(y) = 0 (21)$$

211
$$C_1''(y) - (c_1 i\omega + c_2)C_1(y) = 0$$

212 Solving equations (21) and (22) and imposing the boundary conditions of equation (17)

(22)

213 as well as substituting in equation (20), we get

214
$$C(y,t) = Sinh\sqrt{c_2}y + \varepsilon \exp(i\omega t)Sinh\sqrt{\beta}y$$
 (23)

- where $\beta = c_1 i\omega + c_2$
- Similarly, we put equation (19) into equation (13) and the resulting expression takes the
- 217 form

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219
$$\theta_0''(y) - b_2 \theta_0(y) = 0$$
 (24)

220
$$\theta_1''(y) - (b_1 i\omega + b_2)\theta_1(y) = 0$$
 (25)

- The solutions of equations (24) and (25) and imposition of the boundary conditions of
- equation (16) as well as put the entire expression into equation (19), gives

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224
$$\theta(y,t) = Sinh\sqrt{b_2}y + \varepsilon \exp(i\omega t)Sinh\sqrt{\beta_2}y$$
 (26)

225 where $\beta_2 = b_1 i\omega + b_2$

Also, following the same procedure and the boundary conditions of equation (18), the 226

227 solution of equation (12) is

$$u(y,t) = \beta_5 Sinh \sqrt{\frac{a_2}{w_2}} y + \beta_3 Sinh \sqrt{b_2} y + \beta_4 Sinh \sqrt{c_2} y + 228$$

$$\varepsilon \exp(i\omega t) \left(-\beta_8 Cosh \sqrt{\frac{a_1 + a_2}{w_2}} y + \beta_9 Sinh \sqrt{\frac{a_1 + a_2}{w_2}} y + \beta_6 Sinh \sqrt{\beta_2} y + \beta_7 Sinh \sqrt{\beta} y + \beta_8 \right)$$

229 (27)

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231 where,
$$\beta_3 = \frac{-\frac{a_3}{w_2}}{b_2^2 - \frac{a_2}{w_2}}$$
, $\beta_4 = \frac{a_4}{w_2(c_2^2 - 1)}$, $\beta_5 = \frac{-\beta_3 Sinh\sqrt{b_2} - \beta_4 Sinh\sqrt{c_2}}{Sinh\sqrt{\frac{a_2}{w_2}}}$,

232
$$\beta_{6} = \frac{-\frac{a_{3}}{w_{2}}}{\beta_{2} - \frac{a_{1} + a_{2}}{w_{2}}}, \beta_{7} = \frac{-\frac{a_{4}}{w_{2}}}{\beta - \frac{a_{1} + a_{2}}{w_{2}}}, \beta_{8} = \frac{\lambda}{a_{1} + a_{2}},$$

$$\beta_{9} = \frac{-\left(\beta_{8} + \beta_{6} Sinh\sqrt{\beta_{2}} + \beta_{7} Sinh\sqrt{\beta}\right)}{Sinh\sqrt{\frac{a_{1} + a_{2}}{w}}}$$

233
$$\beta_9 = \frac{-\left(\beta_8 + \beta_6 Sinh\sqrt{\beta_2} + \beta_7 Sinh\sqrt{\beta}\right)}{Sinh\sqrt{\frac{a_1 + a_2}{w}}}$$

Case 2: The upper wall of the channel is set into oscillatory motion while the lower wall 234

235 is held stationary and the resulting boundary conditions are given as

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237
$$u(0,t) = 0$$
, and $u(1,t) = H(t)\varepsilon \exp(i\omega t)$; $t > 0$ (28)

where H(t) is the Heaviside step function 238

239 Imposing the boundary conditions of equation (28) into the solution of equation (12), we

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241

$$u(y,t) = \left(\frac{H(t)\varepsilon \exp(i\omega t) - \beta_3 Sinh\sqrt{b_2} - \beta_4 Sinh\sqrt{c_2}}{Sinh\sqrt{\frac{a_2}{w_2}}}\right) Sinh\sqrt{\frac{a_2}{w_2}}y$$

$$242 + \beta_3 Sinh\sqrt{b_2}y + \beta_4 Sinh\sqrt{c_2}y$$

$$+ \varepsilon \exp(i\omega t) \left[\frac{H(t)\varepsilon \exp(i\omega t) - \beta_6 Sinh\sqrt{\beta_2} - \beta_7 Sinh\sqrt{\beta} - \beta_8}{Sinh\sqrt{\frac{a_1 + a_2}{w_2}}} \right] Sinh\sqrt{\frac{a_1 + a_2}{w_2}} y + \beta_{10}$$

243 (29)

- where $\beta_{10} = \beta_6 Sinh\sqrt{\beta_2} y + \beta_7 Sinh\sqrt{\beta} y + \beta_8$
- Case 3: In this situation, the two channel walls are set into oscillatory motion and the resulting boundary conditions (Aiza et al (2105)) are given as
- 249 $u(0,t) = H(t)\varepsilon \exp(i\omega t)$ and $u(1,t) = H(t)\varepsilon \exp(i\omega t)$ t > 0 (30)
- 250 The solution of equation (12), having used the boundary conditions of equation (30),
- 251 following the same procedure used in case 2, we obtain

$$u(y,t) = H(t)\varepsilon \exp(i\omega t)Cosh\sqrt{\frac{a_2}{w_2}}y +$$

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$$\left(\frac{H(t)\varepsilon\exp(i\omega t) - \beta_{3}Sinh\sqrt{b_{2}} - \beta_{4}Sinh\sqrt{c_{2}}}{Sinh\sqrt{\frac{a_{2}}{w_{2}}}}\right)Sinh\sqrt{\frac{a_{2}}{w_{2}}}y$$

- $+ \beta_3 Sinh\sqrt{b_2} y + \beta_4 Sinh\sqrt{c_2} y$
- $+H(t)\varepsilon \exp(i\omega t)Cosh\sqrt{\frac{a_1+a_2}{w_2}}y+$

$$\left(\frac{H(t)\varepsilon\exp(i\omega t) - \beta_6 Sinh\sqrt{\beta_2} - \beta_7 Sinh\sqrt{\beta} - \beta_8}{Sinh\sqrt{\frac{a_1 + a_2}{w_2}}}\right) Sinh\sqrt{\frac{a_1 + a_2}{w_2}}y$$

- $253 + \beta_6 Sinh\sqrt{\beta_2} y + \beta_7 Sinh\sqrt{\beta} y + \beta_8$ (31)
- Nusselt number (Nu) (the rate of heat transfer coefficient)
- From equation (26), the Nusselt number is given by

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$$Nu = -\left(\frac{\partial \theta}{\partial y}\right)_{y=0} = -\left(\sqrt{b_2} + \varepsilon \exp(i\omega t)\sqrt{\beta_2}\right)$$
 (32)

259 **Sherwood number** (S_b) (the rate of mass transfer coefficient)

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$$S_b = -\left(\frac{\partial C}{\partial y}\right)_{y=0} = -\left(\sqrt{C_2} + \varepsilon \exp(i\omega t)\sqrt{\beta}\right)$$
 (33)

261 Skin frictions (τ)

$$262 \qquad \left(\frac{\partial u}{\partial y}\right)_{y=0} = \beta_5 \sqrt{\frac{a_2}{w_2}} + \beta_3 \sqrt{b_2} + \beta_4 \sqrt{c_2} + \varepsilon \exp(i\omega t) \left(\beta_9 \sqrt{\frac{a_1 + a_2}{w_2}} + \beta_6 \sqrt{\beta_2} + \beta_7 \sqrt{\beta}\right) (34)$$

263 Case 2

$$\left(\frac{\partial u}{\partial y}\right)_{y=0} = \left(\frac{H(t)\varepsilon\exp(i\omega t) - \beta_{3}Sinh\sqrt{b_{2}} - \beta_{4}Sinh\sqrt{c_{2}}}{Sinh\sqrt{\frac{a_{2}}{w_{2}}}}\right)\sqrt{\frac{a_{2}}{w_{2}}} + \beta_{3}\sqrt{b_{2}} + \beta_{4}\sqrt{c_{2}}$$

$$+ \varepsilon\exp(i\omega t)\left(\frac{H(t)\varepsilon\exp(i\omega t) - \beta_{6}Sinh\sqrt{\beta_{2}} - \beta_{7}Sinh\sqrt{\beta} - \beta_{8}}{Sinh\sqrt{\frac{a_{1} + a_{2}}{w_{2}}}}\right)\sqrt{\frac{a_{1} + a_{2}}{w_{2}}} + \beta_{77}\right)$$
265 where $\beta_{77} = \beta_{6}\sqrt{\beta_{2}} + \beta_{7}Sinh\sqrt{\beta}$

where $\beta_{77} = \beta_6 \sqrt{\beta_2} + \beta_7 Sinh \sqrt{\beta}$ 265

Case 3

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$$\left(\frac{\partial u}{\partial y}\right)_{y=0} = \left(\frac{H(t)\varepsilon\exp(i\omega t) - \beta_{3}Sinh\sqrt{b_{2}} - \beta_{4}Sinh\sqrt{c_{2}}}{Sinh\sqrt{\frac{a_{2}}{w_{2}}}}\right)\sqrt{\frac{a_{2}}{w_{2}}} + \beta_{3}\sqrt{b_{2}} + \beta_{4}\sqrt{c_{2}} + \left(\frac{H(t)\varepsilon\exp(i\omega t) - \beta_{6}Sinh\sqrt{\beta_{2}} - \beta_{7}Sinh\sqrt{\beta} - \beta_{8}}{Sinh\sqrt{\frac{a_{1} + a_{2}}{w_{2}}}}\right)\sqrt{\frac{a_{1} + a_{2}}{w_{2}}} + \beta_{6}\sqrt{\beta_{2}} + \beta_{7}\sqrt{\beta}$$
(36)

4. Discussion

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- As a result of increase in the chemical reaction of the nanoparticles and the base fluid, a corresponding increase in the concentration of the nanofluid is observed as depicted in Figure 1
- Figure 2, showed the effect of radiation on the temperature of copper nanofluid. As a result of high thermal conductivity of nanofluid, increase in radiation of nanofluid result in a corresponding increase in the temperature profile of the nanofluid. This observation is consistent with the work of Aaiza et al (2015) and Timofeeva et al (2009) and a departure from the effect of radiation on other base fluids.

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Figure 3, showed the convective heat transfer coefficient between the nanofluid and the channel walls to estimate the heat transfer performance of the nanofluid. The graph showed that, the heat transfer is rapid (a curve), may be due to the presence of radiation when compared to the work of Xuan and Li (2000) which is a straight line. The peclet number is a parameter to describe such effect. Equation (32) therefore, is a holistic approach to study the improved heat transfer mechanism of the nanofluid.

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From Figure 4, it is shown that the relationship can be used to determine the convective mass transfer coefficient between the nanofluid and the containing inner wall to estimate the improved mass transfer performance of the nanoparticles.

Figure 5, clearly demonstrate that, increase in peclet number correspond to a decrease in the skin friction of the nanofluid.

Increase in nanoparticles volume fraction of copper nanofluid result in a corresponding increase in the velocity profile of the nanofluid as shown in Figure 6.

increase in the velocity profile of the nanofluid as shown in Figure 6.
Reynolds number describe the transition of fluids from laminar to turbulence, since it is a ratio of inertia force to viscous force, its increase as shown in Figure 7, results in a decrease in the velocity profile of nanofluid and this observation is at variance with the results of increase in Reynolds number of base fluids (Ngiangia and Taylor-Harry 2017). Electroconductivity tends to impede the flow of fluid and same observation was noticed as depicted in Figure 8, where increase in electroconductivity enhances the decrease in the velocity of copper nanofluid.

The Grashof number due to temperature conducts heat away from the channel plates into the nanofluid, thereby increasing the temperature of the fluid but as a result of rapid heat transfer and thermal conductivity of nanofluid, a decrease in velocity profile of the nanofluid is observed as shown in Figure 9.

Figure 10, showed the effect of different shapes of copper nanoparticles on the velocity of water based nanofluids. The graph clearly depicts that, the highest velocity is recorded with brick followed by cylinder, platelet and finally blade. The effect on shapes of nanoparticles on velocity is also a dependence on viscosity for various volume fractions.

The following numerical values are used in the plotting of the graphs $\lambda_n = 100, \omega = 0.2, t = 1, \lambda = 1, \varepsilon = 0.003, k = 0.3$ $Gr_T, Gr_C = 1,3,5,7$ Pe = 0.35,0.70,1.05,1.40 $\phi = 0.1,0.2,0.3,0.4$ Re = 100,1500,2000,2500 N = 2.0,4.0,6.0,8.0

 $k_{\infty} = 1.35, 2.35, 3.35, 4.45$

 $\sigma_0 = 0.4, 0.8, 1.2, 1.6$

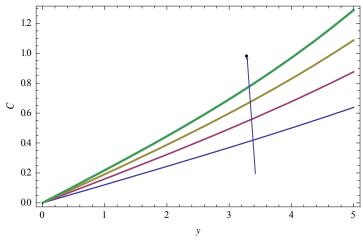


Fig.1. The dependence of concentration on coordinate with chemical reaction $k_{\scriptscriptstyle \infty}$ varying

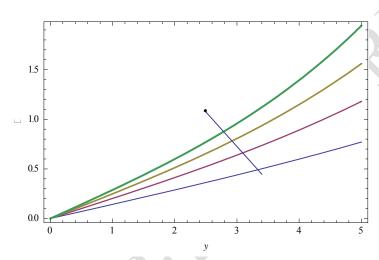


Fig.2. The dependence of temperature on coordinate with radiation ${\cal N}$ varying

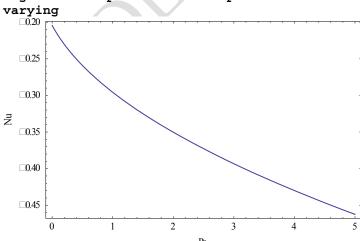


Fig.3. The Nusselt number(Nu) versus Peclet number(Pe)



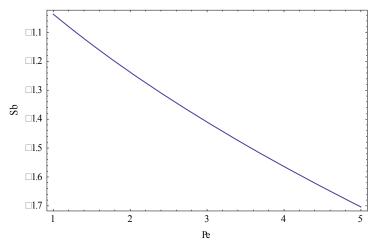
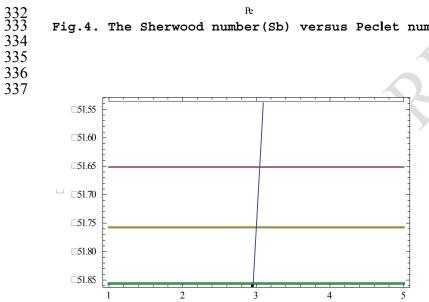


Fig.4. The Sherwood number(Sb) versus Peclet number(Pe)



Re

Fig.5. The Skin Friction(τ) versus Reynolds number(Re)with Peclet number(Pe) varying.

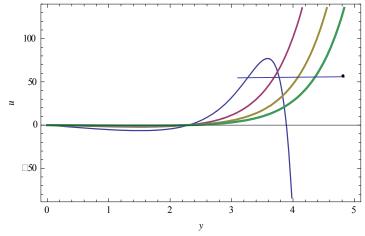


Fig.6. The dependence of velocity on coordinate with nanoparticles volume fractions ϕ varying

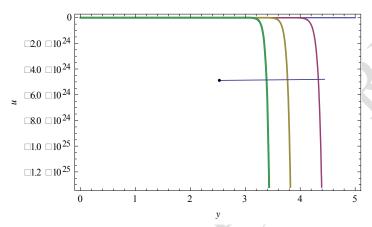


Fig.7. The dependence of velocity on coordinate with $\ensuremath{\text{Reynolds}}$ number Re varying

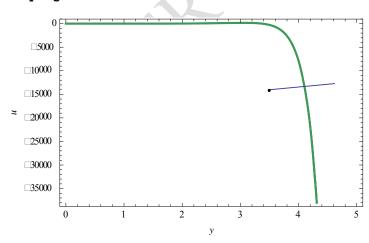


Fig.8. The dependence of velocity on coordinate with conductivity $\boldsymbol{\sigma}_{\boldsymbol{0}}$ varying

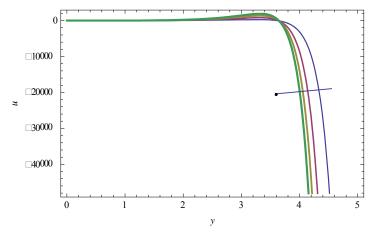


Fig.9. The dependence of velocity on coordinate with Grashof number $Gr_{\scriptscriptstyle T}$ varying

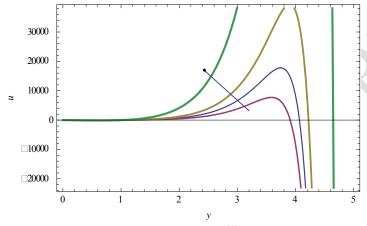


Fig.10. The dependence of velocity on coordinate with different nanoshape particles \boldsymbol{n}

Table 1: Constants **a** and **b** empirical shape factors, Timofeeva et al (2009)

Model	Platelet	Blade	Cylinder	Brick
a	37.1	14.6	13.5	1.9
b	612.6	123.3	904.4	471.4

Table 2: Sphericity for different shapes nanoparticles, Timofeeva et al (2009)

Model	Platelet	Blade	Cylinder	Brick
Ψ	0.52	0.36	0.62	0.81

Table 3; Thermophysical properties of water and nanoparticle, Timofeeva et al (2009)

Model	$\rho(kgm^{-3})$	$C_p(kg^{-1}K^{-1})$	$k(Wm^{-1}K^{-1})$	$\beta x 10^{-5} K^{-1}$
Water (H_2O)	997.1	4179	0.613	21
Copper (Cu)	8933	385	401	1.67

370 **5. Conclusions**

- 371 A perturbation technique on the description of the heat and mass transfer of copper
- anofluid has been carried out. An essential characteristic of nanofluid, in particular
- 373 copper nanoparticles in water based fluids is that, as a result of its high thermal
- 374 conductivity and heat transfer ability, the effect on velocity profile is significantly
- different from other conventional base fluids. Experiments is needed for industrial,
- scientific and engineering applications of copper nanaofluid and others. It should be
- noted that the improved performance of the copper nanofluid is not limited to its high
- 378 thermal conductivity and heat transfer ability but also from the random movement and
- dispersion effect on the nanoparticles. The effect of Grashof number on the concentration
- of nanofluid was ignored because the result is the same as that of the temperature.

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