

Natural Convection Couette Flow through a Vertical Porous Channel Due to Combined Effects of Thermal Radiation and Variable fluid Properties

Abstract:

The present article investigates natural convection Couette flow through a vertical porous channel due to combined effects of thermal radiation and variable fluid properties. The fluid considered in the model is of an optically dense with all its physical properties assumed constant except for its viscosity and thermal conductivity which are temperature dependent. The flow equations are simplified using non-linear Rosseland heat diffusion and as a consequence it resulted to high non-linearity of the flow equations. Adomian decomposition method (ADM) is used to solve the emanating equations and the influences of the essential controlling physical parameters involved are presented on graphs, tables and were discussed. In the course of investigation; it was found that both the fluid velocity and its temperature within the channel were seen to increase with growing thermal radiation parameter while the fluid's velocity and temperature were observed to descend with increase in thermal conduction of the fluid. Similarly; the fluid velocity was found to increase with decrease in the fluid viscosity. To validate the accuracy of the present investigation; the result obtained herein has been compared with a published work where good agreement was found.

Key words: Couette flow; Variable viscosity; Variable thermal conductivity, Thermal radiation; Porous channel; ADM.

1.0 Introduction:

Couette flow is a phenomenon in fluid flow which occurs due to the movement of bounding surface surrounding the fluid. This type of flow occurs in fluid machinery involving moving parts; especially in hydrodynamics lubrication and it has been used as the fundamental method for measurement of viscosity and as a means of estimating the drag force in many wall driven applications (Yasutomi [1]).

Study of fluid flows with variable viscosity through porous media has been conducted by many researchers in view of its application in sciences and engineering; particularly in the utilization of geothermal energy, high performance building insulation, crude oil extraction, petroleum industries, solid matrix exchangers, chemical catalytic reactor, underground disposal of nuclear waste materials and many others. A comprehensive survey in relation to the above applications has been given by Neild and Bejan [2] and Pop and Ingham [3]. In industrial systems, fluid can be subjected to extreme conditions such as high temperature, pressure and shear rate; external heating such as high shear rate can lead to high temperature being generated within the fluid and as a consequence affect its viscosity (Macosco [4]). The usual assumption of constant viscosity property of fluids evaluated at some reference temperature is not enough to depict a true situation in the flow characteristics of boundary layer flows. In view thereof, Mehta and Sood [5] and Garey *et al.* [6] submitted that when varying viscosity property of fluid is included, the flow characteristics change substantially compared to the constant case. Related literature can be viewed in Kafousius and Williams [7] and Kafousius and Rees [8]. The latter researchers concluded that when viscosity of the fluid is sensitive to temperature variation, the effect of temperature-dependent viscosity has to be taken into consideration; otherwise considerable errors may occur in the heat transfer characteristics.

Boundary layer flows with temperature dependent thermal conductivity has been studied by investigators owing to its applications in engineering technology as in the extrusion of plastic sheets, polymer processing, spinning of fibers, cooling of elastic sheets etc. The quality of final products in manufacturing industries relies on the rate of heat transfer and therefore cooling procedure has to be controlled effectively. Materials of high thermal conductivity are widely used in heat sink/source applications while those of low conductivity are used as insulators. Liquid metals that have small Prandtl number in the range 0.01 – 0.1 are generally used as coolants because of their high thermal conductivity. The studies of Van den Berg *et al.* [9] has shown that variable thermal conductivity can delay secular cooling of mantle with constant viscosity model while that of Sharma and Rafi [10] disclosed that thermal conduction increases with decrease in Prandtl number. Other related results can be observed in Starlin *et al.* [11], Dubuffet [12], Hofmeister [13] and van den Berg *et al.* [14].

Entropy generation is a measure of destruction of the available work done by a system; when it occurs, the consequent effect is the emission of heat in the form of electromagnetic rays termed as “thermal radiation”. Radiation effects on free convection flows are important in the context of space technology and processes involving high temperature, this is due to the safety of lives and properties especially in working conditions that requires liberation of heat. In order to minimize entropy generation/thermal radiation; several studies were carried out to this effect; notable

among them are Ajibade *et al.* [15], Makinde [16], Ibanez *et al.* [17] and Schlichting and Mahmud [18]. The study of Makinde and Ogulu [19] analyzed the effect of thermal radiation on heat and mass transfer flow of variable viscosity fluid past a vertical porous plate permeated by a transverse magnetic field where they concluded that an increase in the positive value of viscosity variation parameter decreases the fluid viscosity and consequently leads to an increase in the velocity within the boundary layer while Hossain *et al.* [20] studied the effect of radiation on free convective flow of fluid with variable viscosity from a porous vertical plate. Other similar researches can be seen in Costa and Macedonio [21], Seddeek and Salem [22], Elbasbeshy and Bazid [23], Makinde and Ibrahim [24] and Makinde *et al.* [25]. Rosseland [26] gave an expression for radiative heat flux which was later simplified by Sparrow and Cess. [27] and this is widely used in the studies of heat transfer with thermal radiation by scholars with success. In realization of this notable achievement; related studies can be seen in [19], [24] and [25] where they discussed the effect of thermal radiation on boundary layer flows using linearized Rosseland heat diffusion. This was however faulted by Magyari and Pantokratoras [28] on the basis that it does not reflect the real mechanism in heat transfer characteristics in most boundary layer flows with thermal radiation and they therefore gave alternative approach using non-linear Rossland heat diffusion.

This paper investigates natural convection Couette flow through a vertical porous channel due to combined effects of thermal radiation and variable fluid properties. In particular; the variable fluid properties considered are that, the viscosity and thermal conductivity assumed variable status following Carey and Mollendorf [29] with the radiative heat flux adopting non-linear Rossland heat diffusion.

2.0 Mathematical formulation:

The schematic diagram below in figure 1 consists of an infinite vertical channel formed by two parallel plates stationed h distance apart. The channel is filled with an optically dense viscous incompressible fluid at the expense of radiative heat flux of intensity q_r ; which is absorbed by the plates and transferred to the fluid. The fluid's physical properties are assumed constant except for its viscosity and thermal conductivity which are temperature dependent. Since the fluid is an optically dense; the radiative heat flux of Rosseland heat diffusion can be utilized to analyze the energy equation in the flow formation. The x' -axis is taken along the channel in the vertically upward direction, being the direction of the flow while the y' -axis is taken normal to it and the effect of radiative heat flux in the x' - direction is considered negligible compared to that in the y' - direction. The temperature of the plate kept at $y' = 0$ rise to T_w and thereafter maintained constant while the other plate at $y' = h$ remain at T_0 . Also, the plate at $y' = 0$ moves at its own plate impulsively at uniform velocity $u' = Mu_0$ while the other plate remains at rest.

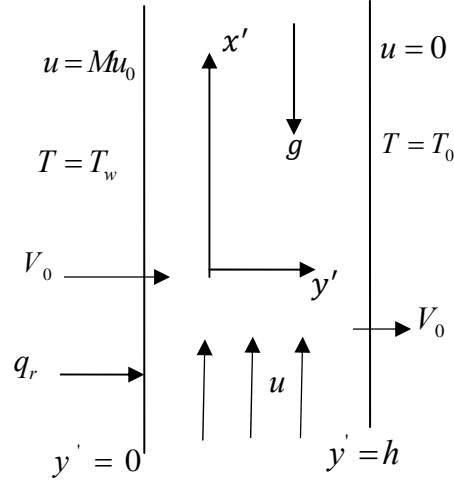


Fig 1: Schematic diagram of the problem

Under these assumptions; the appropriate governing equations are:

$$V_0 \frac{\partial u'}{\partial y'} + \frac{1}{\rho} \frac{\partial}{\partial y'} \left(\mu \frac{\partial u'}{\partial y'} \right) + g\beta(T - T_0) = 0 \quad (1)$$

$$V_0 \frac{\partial T'}{\partial y'} + \frac{1}{\rho c_p} \frac{\partial}{\partial y'} \left(k \frac{\partial T'}{\partial y'} \right) - \frac{1}{\rho c_p} \frac{\partial q_r}{\partial y'} = 0 \quad (2)$$

with the radiative heat flux q_r in the form:

$$q_r = \frac{-4\sigma \partial T'^4}{3\delta \partial y'} \quad (\text{Sparrow and Cess. [27]}) \quad (3)$$

Following Carey and Mollendorf [29]; the fluid viscosity and thermal conductivity respectively are of the form:

$$\mu = \mu_0 \left(1 - \lambda \left(\frac{T - T_0}{T_w - T_0} \right) \right), k = k_0 \left(1 - \varepsilon \left(\frac{T - T_0}{T_w - T_0} \right) \right) \quad \text{for } \lambda, \varepsilon \in \Re \quad (4)$$

and the initial and boundary conditions for the velocity and temperature fields as:

$$u' = 0, T' = T_0 \quad \text{for } 0 \leq y' \leq h \quad (5)$$

$$\begin{cases} u' = Mu_0, T' = T_w & \text{at } y' = 0 \\ u' = 0, T' = T_0 & \text{at } y' = h \end{cases} \quad (6)$$

The following dimensionless quantities are introduced:

$$u = \frac{u'}{u_0}, y = \frac{y'}{h}, \theta(y) = \frac{T' - T_0}{T_w - T_0} \quad (7)$$

Using equations (3), (4) and (7) in equation (1), the equation for the velocity field in dimensionless form is:

$$u''(y) = -c(1 + \lambda\theta(y))u'(y) + \lambda(1 + \lambda\theta(y))\theta'(y)u'(y) - Gr\theta(y)(1 + 2\lambda\theta(y)) \quad (8)$$

Following Magyari and Pantokratoras [28] and using equation (7); $\frac{\partial q_r}{\partial y'}$ is expanded as follow:

$$\begin{aligned} \frac{\partial q_r}{\partial y'} &= \frac{\partial}{\partial y'} \left[\left(-\frac{4\sigma}{3\delta} \right) \frac{\partial T'^4}{\partial y'} \right] = -\frac{4\sigma}{3h^2\delta} \left(\frac{\partial^2}{\partial y'^2} \left([\theta(y)(T_w - T_0) + T_0]^4 \right) \right) \\ &= -\frac{4\sigma}{3h^2\delta} \left(\frac{\partial}{\partial y'} \left(4[\theta(y)(T_w - T_0) + T_0]^3 \right) \frac{\partial}{\partial y'} (\theta(y)(T_w - T_0)) \right) \\ &= -\frac{4\sigma}{3h^2\delta} \left(12[\theta(y)(T_w - T_0) + T_0]^2 \frac{\partial}{\partial y'} (\theta(y)(T_w - T_0)) \frac{\partial}{\partial y'} (\theta(y)(T_w - T_0)) \right) \\ &\quad - \frac{4\sigma}{3h^2\delta} \left(4[\theta(y)(T_w - T_0) + T_0]^3 \frac{\partial^2}{\partial y'^2} (\theta(y)(T_w - T_0)) \right) \\ &= -\frac{4\sigma}{3h^2\delta} \left(12(T_w - T_0)^4 [\theta(y) + \phi]^2 \theta'(y)\theta'(y) \right) \\ &\quad - \frac{4\sigma}{3h^2\delta} \left(4(T_w - T_0)^4 [\theta(y) + \phi]^3 \theta''(y) \right) \end{aligned} \quad (9)$$

Substituting equation (3), (4), (7) and (9) in equation (2) and rearranging, the following equation is achieved:

$$\theta''(y) = -c\theta'(y)(1 + \varepsilon\theta(y)) \left[1 - \frac{4R_T}{3}(1 + \varepsilon\theta(y))[\theta(y) + \phi]^3 \right]$$

$$-4R_T(1+\varepsilon\theta(y))\theta'^2(y)[\theta(y)+\phi]^2\left[1-\frac{4R_T}{3}(1+\varepsilon\theta(y))[\theta(y)+\phi]^3\right] \quad (10)$$

Again; on using equation (7) in equations (5 – 6), the initial and boundary conditions are now:

$$u = 0, \theta = 0 \quad \text{for} \quad 0 \leq y \leq 1 \quad (11)$$

$$\begin{cases} u = M, \theta = 1 & \text{at} \quad y = 0 \\ u = 0, \theta = 0 & \text{at} \quad y = 1 \end{cases} \quad (12)$$

$$\text{Where} \quad R_T = \frac{4\sigma(T_w - T_0)^3}{3k_0 \delta}, \quad \phi = \frac{T_0}{T_w - T_0}, \quad c = \frac{hV_0}{\alpha} \quad (13)$$

2.1 ADM Solution of the Problem:

Equations (8) and (9) subject to the boundary conditions (11) and (12) are solved by ADM as follow:

Denote by $u''(y) = \frac{d^2u(y)}{dy^2}$ and let $L = \frac{d^2u(y)}{dy^2}$ so that

$$Lu(y) = u''(y) \quad \text{and} \quad L\theta(y) = \theta''(y) \quad (14)$$

Using equation (15); equations (8) and (10) can be written as:

$$Lu(y) = -c(1+\lambda\theta(y))u'(y) + \lambda(1+\lambda\theta(y))\theta'(y)u'(y) - Gr\theta(y)(1+2\lambda\theta(y)) \quad (15)$$

$$\begin{aligned} L\theta(y) = & -c\theta'(y)(1+\varepsilon\theta(y))\left[1-\frac{4R_T}{3}(1+\varepsilon\theta(y))[\theta(y)+\phi]^3\right] \\ & + \varepsilon L^{-1}\left\{\theta'^2(y)(1+\varepsilon\theta(y))\left[1-\frac{4R_T}{3}(1+\varepsilon\theta(y))[\theta(y)+\phi]^3\right]\right\} \\ & - 4R_T L^{-1}\left\{(1+\varepsilon\theta(y))\theta'^2(y)[\theta(y)+\phi]^2\left[1-\frac{4R_T}{3}(1+\varepsilon\theta(y))[\theta(y)+\phi]^3\right]\right\} \quad (16) \end{aligned}$$

Operating L^{-1} both sides of equations (14) and (15) we obtain:

$$L^{-1}Lu(y) = -cL^{-1} \left\{ (1 + \lambda\theta(y))u'(y) \right\} + \lambda L^{-1} \left\{ (1 + \lambda\theta(y))\theta'(y)u'(y) \right\} \\ - GrL^{-1} \left\{ \theta(y)(1 + 2\lambda\theta(y)) \right\} \quad (17)$$

$$L^{-1}L\theta(y) = -cL^{-1} \left\{ \theta'(y)(1 + \varepsilon\theta(y)) \left[1 - \frac{4R_T}{3}(1 + \varepsilon\theta(y))[\theta(y) + \phi]^3 \right] \right\} \\ + \varepsilon L^{-1} \left\{ \theta'^2(y)(1 + \varepsilon\theta(y)) \left[1 - \frac{4R_T}{3}(1 + \varepsilon\theta(y))[\theta(y) + \phi]^3 \right] \right\} \\ - 4R_T L^{-1} \left\{ (1 + \varepsilon\theta(y))\theta'^2(y)[\theta(y) + \phi]^2 \left[1 - \frac{4R_T}{3}(1 + \varepsilon\theta(y))[\theta(y) + \phi]^3 \right] \right\} \quad (18)$$

Where $L^{-1} = \int \int (\bullet) dy dy$

By ADM:

$$L^{-1}Lu(y) = u(y) - u(0) - yu'(0) \quad (19)$$

$$L^{-1}L\theta(y) = \theta(y) - \theta(0) - y\theta'(0) \quad (20)$$

Using equation (11), (19) and (20) in equations (17) and (18) we have:

$$u(y) = M + yA - cL^{-1} \left\{ (1 + \lambda\theta(y))u'(y) \right\} + \lambda L^{-1} \left\{ (1 + \lambda\theta(y))\theta'(y)u'(y) \right\} \\ - GrL^{-1} \left\{ \theta(y)(1 + 2\lambda\theta(y)) \right\} \quad (21)$$

$$\theta(y) = 1 + yB - cL^{-1} \left\{ \theta'(y)(1 + \varepsilon\theta(y)) \left[1 - \frac{4R_T}{3}(1 + \varepsilon\theta(y))[\theta(y) + \phi]^3 \right] \right\} \\ + \varepsilon L^{-1} \left\{ \theta'^2(y)(1 + \varepsilon\theta(y)) \left[1 - \frac{4R_T}{3}(1 + \varepsilon\theta(y))[\theta(y) + \phi]^3 \right] \right\} \\ - 4R_T L^{-1} \left\{ (1 + \varepsilon\theta(y))\theta'^2(y)[\theta(y) + \phi]^2 \left[1 - \frac{4R_T}{3}(1 + \varepsilon\theta(y))[\theta(y) + \phi]^3 \right] \right\} \quad (22)$$

Where $A = f'(0)$ and $B = \theta'(0)$ are assumed values to be determined based on the boundary condition in equation (12).

According to standard ADM, $u(y)$ and $\theta(y)$ can be expressed in the forms:

$$u(y) = \sum_{n=0}^{\infty} u_n(y) \quad \text{and} \quad \theta(y) = \sum_{n=0}^{\infty} \theta_n(y) \quad (23)$$

Using equation (23) in equations (21) and (22), we have:

$$\begin{aligned} \sum_{n=0}^{\infty} u_n(y) = & M + yA - cL^{-1} \left\{ \left(1 + \lambda \sum_{n=0}^{\infty} \theta_n(y) \right) \frac{d}{dy} \left(\sum_{n=0}^{\infty} u_n(y) \right) \right\} \\ & + \lambda L^{-1} \left\{ \left(1 + \lambda \sum_{n=0}^{\infty} \theta_n(y) \right) \frac{d}{dy} \sum_{n=0}^{\infty} (\theta_n(y)) \frac{d}{dy} \sum_{n=0}^{\infty} (u_n(y)) \right\} \\ & - GrL^{-1} \left\{ \sum_{n=0}^{\infty} \theta_n(y) \left(1 + 2\lambda \sum_{n=0}^{\infty} \theta_n(y) \right) \right\} \end{aligned} \quad (24)$$

$$\begin{aligned} \sum_{n=0}^{\infty} \theta_n(y) = & 1 + yB - cL^{-1} \left\{ \frac{d}{dy} \left(\sum_{n=0}^{\infty} \theta_n(y) \right) \left(1 + \varepsilon \sum_{n=0}^{\infty} \theta_n(y) \right) \left[1 - \frac{4R_T}{3} \left(1 + \varepsilon \sum_{n=0}^{\infty} \theta_n(y) \right) \left[\sum_{n=0}^{\infty} \theta_n(y) + \phi \right]^3 \right] \right\} \\ & + \varepsilon L^{-1} \left\{ \frac{d^2}{dy^2} \left(\sum_{n=0}^{\infty} \theta_n(y) \right) \left(1 + \varepsilon \sum_{n=0}^{\infty} \theta_n(y) \right) \left[1 - \frac{4R_T}{3} \left(1 + \varepsilon \sum_{n=0}^{\infty} \theta_n(y) \right) \left[\sum_{n=0}^{\infty} \theta_n(y) + \phi \right]^3 \right] \right\} \\ & - 4R_T L^{-1} \left\{ \left(1 + \varepsilon \sum_{n=0}^{\infty} \theta_n(y) \right) \left[\sum_{n=0}^{\infty} \theta_n(y) + \phi \right]^2 \frac{d^2}{dy^2} \left(\sum_{n=0}^{\infty} \theta_n(y) \right) \left[1 - \frac{4R_T}{3} \left(1 + \varepsilon \sum_{n=0}^{\infty} \theta_n(y) \right) \left[\sum_{n=0}^{\infty} \theta_n(y) + \phi \right]^3 \right] \right\} \end{aligned} \quad (25)$$

$$\text{Setting } \theta_0(y) = 1 + By \text{ and } u_0(y) = M + yA - GrL^{-1} \left\{ (1 + 2\lambda\theta(y))\theta(y) \right\} \quad (26)$$

then $u_{n+1}(y)$ and $\theta_{n+1}(y)$ for $n \geq 0$ are determined using the recursive relations:

$$u_{n+1}(y) = -cL^{-1} \left\{ \left(1 + \lambda \theta_n(y) \frac{d}{dy} (u_n(y)) \right) \right\} + \lambda L^{-1} \left\{ \left(1 + \lambda \theta_n(y) \right) \frac{d}{dy} (\theta_n(y)) \frac{d}{dy} (u_n(y)) \right\} \quad (27)$$

$$\begin{aligned} \theta_{n+1}(y) = & -cL^{-1} \left\{ \frac{d}{dy} (\theta_n(y)) (1 + \varepsilon \theta_n(y)) \left[1 - \frac{4R_T}{3} (1 + \varepsilon \theta_n(y)) [\theta_n(y) + \phi] \right]^3 \right\} \\ & + \varepsilon L^{-1} \left\{ \frac{d}{dy} (\theta_n(y)) \frac{d}{dy} (\theta_n(y)) (1 + \varepsilon \theta_n(y)) \left[1 - \frac{4R_T}{3} (1 + \varepsilon \theta_n(y)) [\theta_n(y) + \phi] \right]^3 \right\} \\ & - 4R_T L^{-1} \left\{ (1 + \varepsilon \theta_n(y)) \frac{d^2}{dy^2} (\theta_n(y)) [\theta_n(y) + \phi]^2 \left[1 - \frac{4R_T}{3} (1 + \varepsilon \theta_n(y)) [\theta_n(y) + \phi] \right]^3 \right\} \end{aligned} \quad (28)$$

For details on ADM refer to Adomian [30]

2.2 Convergence of the ADM solution and termination criterion of the problem:

Adomian [30] and Cherrault [31] have discussed intensively on the convergence of ADM. Nevertheless; to verify the convergence of the ADM solution in the present problem; the method of ratio test is deployed. Using computer algebra package the following terms were obtained at

$$y = 0.5, \quad M = 1, \quad c = 0.1, \quad \lambda = 0.1, \quad Gr = 10, \quad \phi = 0.1, \quad R_T = 0.1, \quad \varepsilon = 0.1 \quad \text{as:}$$

$$\theta_0 = 0.5124042184, \theta_1 = -0.1703022291, \theta_2 = 0.0003468323623, \theta_3 = 4.31320421237 \times 10^{-6}$$

$$u_0 = 1.190089496, u_1 = -0.02996349680, u_2 = 0.001461778228, u_3 = -0.00004598585703 \quad (29)$$

and

$$\begin{aligned} \left| \frac{\theta_1}{\theta_0} \right| &= 0.03323591473, & \left| \frac{\theta_2}{\theta_1} \right| &= 0.02036569716, & \left| \frac{\theta_3}{\theta_2} \right| &= 0.01243599122 \\ \left| \frac{u_1}{u_0} \right| &= 0.02517751556, & \left| \frac{u_2}{u_1} \right| &= 0.04878530159, & \left| \frac{u_3}{u_2} \right| &= 0.03145884659 \end{aligned} \quad (30)$$

Numerical values in equation (30) shows that

$$\lim_{j \rightarrow \infty} \left| \frac{f_{j+1}}{f_j} \right| < 1, \text{ for } j \geq 0 \text{ Robert [32]} \quad (31)$$

Hence the ADM solution of the present problem converges. For a meaningful series solution, the series needs to be truncated at a point such that the contribution of any additional term is negligible to the final solution. As such a termination criterion is used such that the series is truncated whenever $|u_i, \theta_i| < \varepsilon$. For the present problem, we have chosen $\varepsilon = 3.5 \times 10^{-4}$. Considering this assumption, the solution for u and θ are thus truncated after the 3rd terms. Due to huge size of the computed ADM solution, the solutions are not displayed here rather they are used for numerical computations for the purpose of discussing the result.

2.3 Nusselt number and Skin friction:

Following Kay [33], the Nusselt number on the channel plates stationed at $y = 0$ and $y = 1$ are respectively evaluated using:

$$Nu_0 = (1 - \varepsilon\theta(y)) \left. \frac{d\theta}{dy} \right|_{y=0} \text{ and } Nu_1 = (1 - \varepsilon\theta(y)) \left. \frac{d\theta}{dy} \right|_{y=1} \quad (32)$$

while the skin frictions on the channel plates are calculated via:

$$\tau_0 = (1 - \lambda\theta(y)) \left. \frac{du(y)}{dy} \right|_{y=0} \text{ and } \tau_1 = (1 - \lambda\theta(y)) \left. \frac{du(y)}{dy} \right|_{y=1} \quad (33)$$

2.4 Results and Discussion:

The present article investigated natural convection Couette flow through a vertical porous channel due to combined effects of thermal radiation and variable fluid properties using nonlinear Rosseland heat diffusion. Computer simulation was conducted on the solution of the flow equations considering the influence of the essential physical parameters involved and the

results are presented in figure 2 – 11 and on tables I - III. For the purpose of discussion of this result, the value of R_T and ε are taken within the range $0 \leq R_T, \varepsilon \leq 1$ because terms associated with them behave as strong heat source/sink and large value of these parameters lead to finite time temperature blow up (Makinde and Chinyoka [34]). Similarly, the values for ϕ , λ and M has been chosen arbitrarily between 0.1 – 3 while that of Gr are selected as 10, 12 and 14.

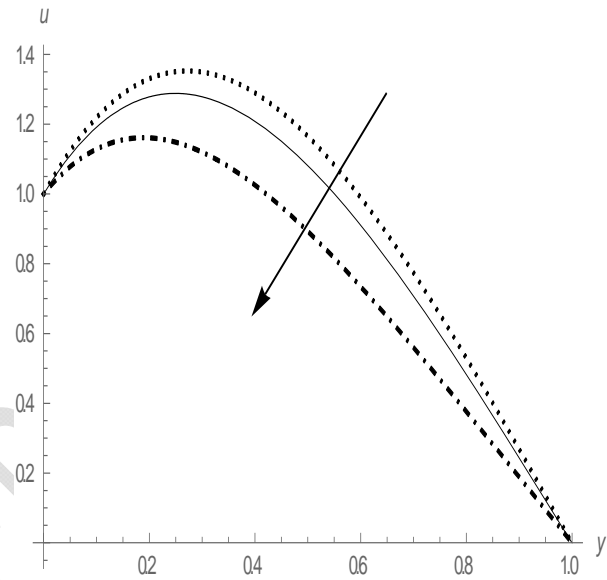
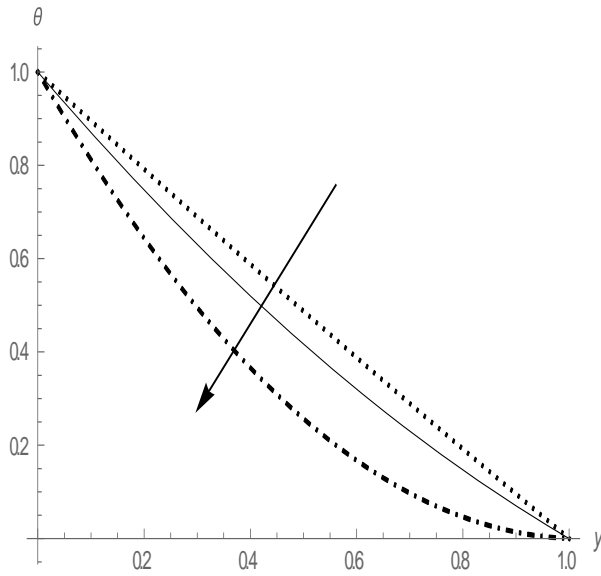


Fig 2: Temperature profiles for different values of c

Fig 3: Velocity profiles for different values of c

($\phi = 0.1, R_T = 0.1, \varepsilon = 0.1, c = \{0.1, 0.5, 1\}$)

($\phi = 0.1, R_T = 0.1, Gr = 10, \lambda = 0.1, M = 0.1,$

$\varepsilon = 0.1, c = \{0.1, 0.5, 1\}$)

Figure 2 and 3 displayed the effects of suction parameter (c) on the velocity and temperature of the fluid within the channel where the figures show that both the temperature and velocity decreases with growing c. These behaviors are accredited to the decrease in thermal diffusivity

of the working fluid with growing c .

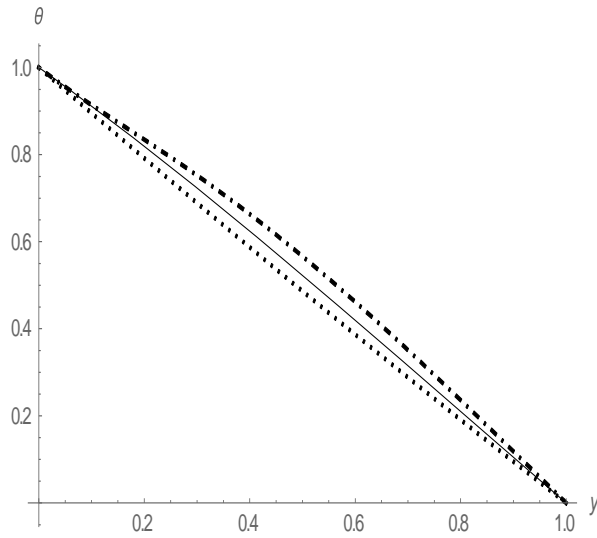


Fig 4: Temperature profiles for different values of R_T
 ($\phi = 0.1, c = 0.1, \epsilon = 0.1, \dots R_T = 0.01, \text{---} R_T = 0.4, \dots R_T = 0.8$)

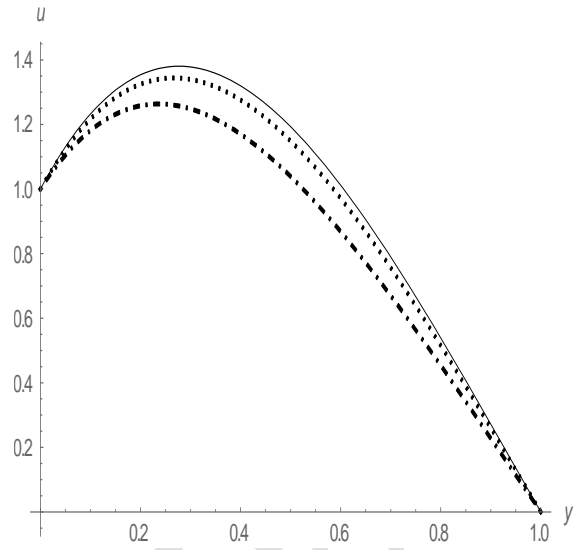


Fig 5: Velocity profiles for different values of R_T
 ($\phi = 0.1, c = 0.1, \lambda = 0.1, Gr = 10, M = 1, \phi = 0.1, \epsilon = 0.1, \dots R_T = 0.001, \dots R_T = 0.4, \text{---} R_T = 0.8$)

The effect of thermal radiation parameter (R_T) on the fluid temperature is naked in figure 4 where the figure shows that the temperature within the channel increases with increase in R_T . The resulting effect of this has transferred to increase the velocity of the fluid within the channel as graphed in figure 5. These attitudes are inclined to the decrease in thermal conduction of the fluid in the channel.

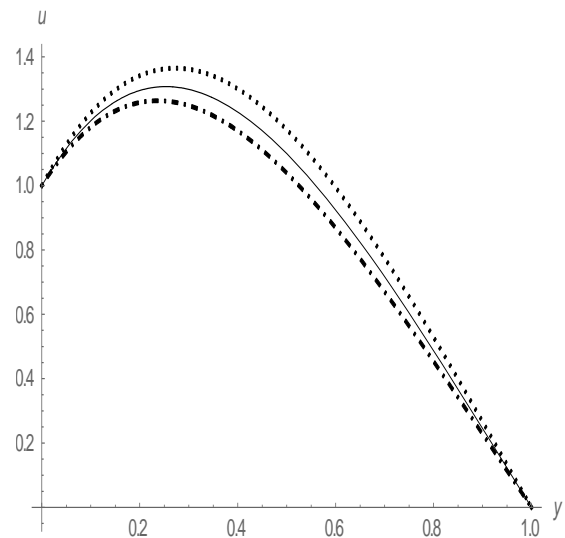
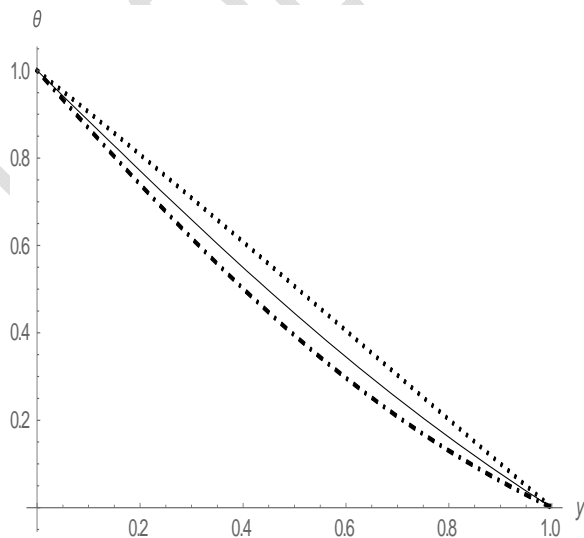


Fig 6: Temperature profiles for different values of ϵ

($\phi = 0.1, c = 0.1, R_T = 0.01, \dots \epsilon = 0.1,$
 $___ \epsilon = 0.4, \dots \epsilon = 0.5$)

Fig 7: Velocity profiles for different values of ϵ

($\phi = 0.1, c = 0.1, \lambda = 0.1, M = 1, R_T = 0.01,$
 $\dots \epsilon = 0.1, ___ \epsilon = 0.4, \dots \epsilon = 0.5$)

Figure 6 and 7 presents the effect of thermal conduction parameter (ϵ) on the velocity and temperature of the fluid within the channel where it is eyed from the figures that, an increase in ϵ contributes to the decrease in the temperature and velocity of the fluid within the channel. This is attributed to the decrease in thermal conduction of the fluid which act to diminish the influence of the applied boundary temperature and thus causing a decrease in the thermodynamics and hydrodynamics of fluid within the channel.

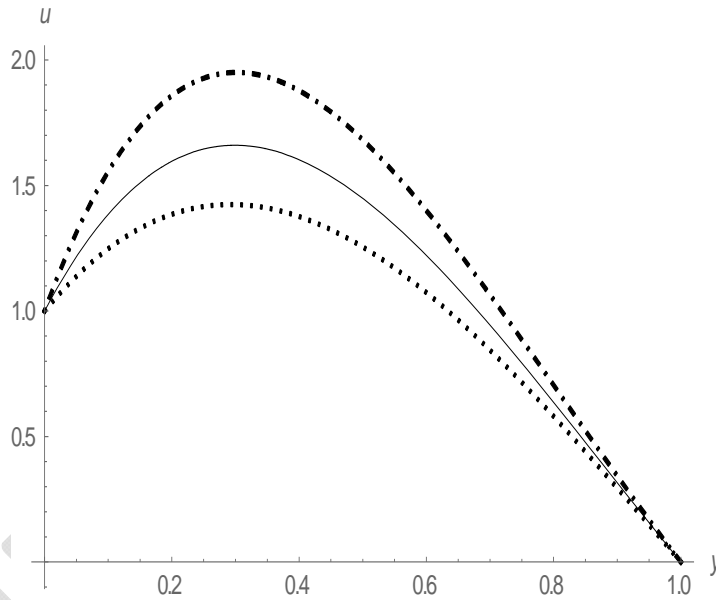


Fig 8: Velocity profile for different values of λ

($\phi = 0.1, c = 0.1, M = 1, Gr = 10, \epsilon = 0.1, R_T = 0.1, \dots \lambda = 0.1, ___ \lambda = 0.4, \dots \lambda = 0.7$)

Figure 8 depicts the influence of viscosity variation parameter (λ) on the fluid velocity in the channel where it is seen that the velocity of the fluid escalate with increase in λ . This is credited to the decrease in the fluid's viscosity within the channel.

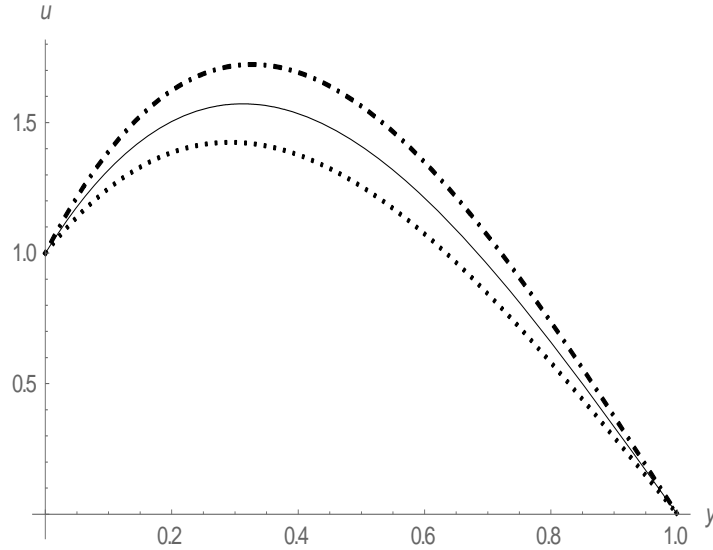


Fig 9: Velocity profile for different values of Gr

($\phi = 0.1, c = 0.1, M = 1, \lambda = 0.1, \epsilon = 0.1, R_T = 0.1, \dots$ Gr = 10, Gr=12, \dots Gr=12)

The effect of varying Gr is pictured in figure 9 where the figure shows that the fluid velocity within the channel increases with ascending Gr. This is accredited to the increase in the buoyancy force of the fluid molecules within the channel and hence an increase in the fluid velocity.

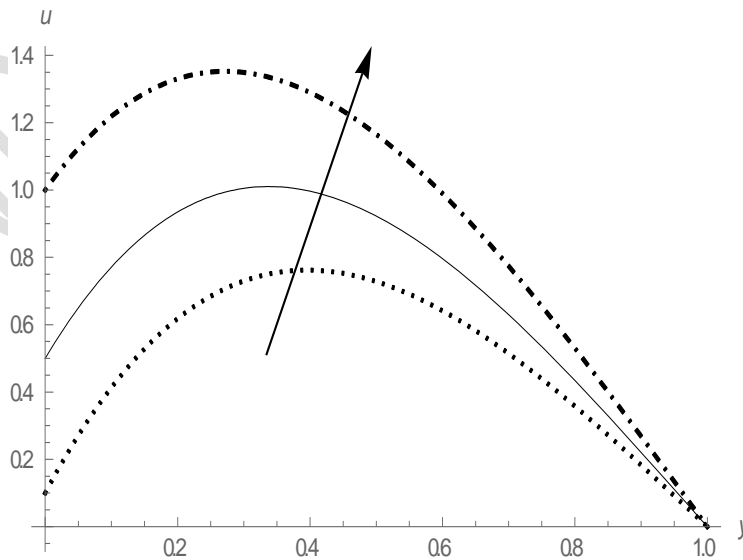


Fig 10: Velocity profile for different values of M

$$(\phi = 0.1, c = 0.1, \lambda = 0.1, Gr = 10, \epsilon = 0.1, R_0 = 0.1, \dots M = 0.1, \text{---} M = 0.5, \text{-.-.-} M = 1)$$

Figure 10 reflects the effect of varying the velocity of the moving boundary on the fluid velocity within the channel. The figure demonstrated that the velocity of the fluid increases with increase M. This is practically true that in no-slip regime, the thin film of the fluid adjacent to the moving plate moves with the velocity of the moving plate.

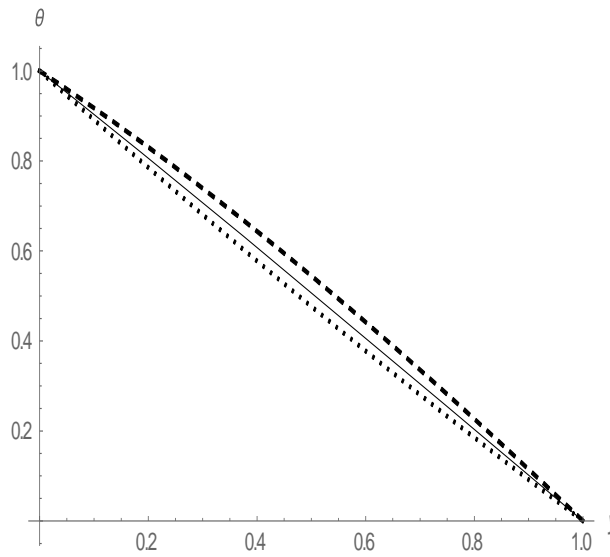


Fig 11: Temperature profile for different values of ϕ ($M = 1, c = 0.1, \lambda = 0.1, Gr = 10, \epsilon = 0.1, R_T = 0.1, \dots \phi = 2, \text{---} \phi = 4, \text{-.-.-} \phi = 6$)

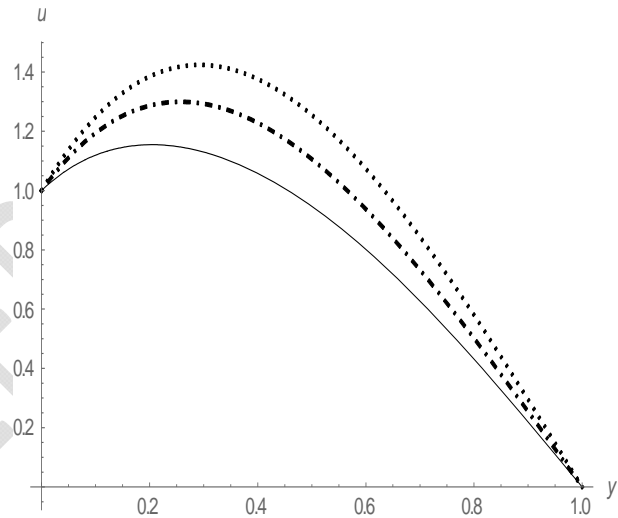


Fig 12: Velocity profile for different values of ϕ ($M = 1, c = 0.1, \lambda = 0.1, Gr = 10, \epsilon = 0.1, R_T = 0.1, \text{---} \phi = 2, \text{-.-.-} \phi = 4, \dots \phi = 6$)

The effect of temperature difference parameter (ϕ) is shown in figure 11 and 12. These figures displayed that the velocity and temperature of the fluid within the channel increases with increase in ϕ . This is owing to the decrease in ambient temperature of the fluid in the channel.

Table I: Nusselt number on the channel plates.

R_T	$\epsilon = 0.01, \phi = 0.1,$ $c = 0.1$	$\epsilon = 0.04, \phi = 0.1,$ $c = 0.1$	$\epsilon = 0.04, \phi = 0.4,$ $c = 0.1$	$\epsilon = 0.04, \phi = 0.4,$ $c = 0.4$
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	Nu_0	Nu_1	Nu_0	Nu_1	Nu_0	Nu_1	Nu_0	Nu_1
0.01	1.03354	0.94317	1.02032	0.92850	1.00723	0.93737	1.1999	0.73108
0.04	1.00392	0.96424	0.97840	0.94748	0.93906	0.97863	1.08756	0.79980
0.06	0.970875	0.97511	0.95426	0.95945	0.90514	1.00234	1.03279	0.83811
0.08	0.95023	0.98557	0.93271	0.97090	0.87807	1.02349	0.98932	0.87162
0.1	0.93166	0.99563	0.91341	0.98184	0.85651	1.04243	0.95456	0.90107

The effects of varying physical parameters on Nusselt numbers is presented on Table I. The table displayed that for some fixed values of ε , ϕ and c ; Nu_0 decreases with increase in R_T while Nu_1 increases with increase in R_T . For small increase in ε ; both Nu_0 and Nu_1 decreases with growth in R_T . Furthermore; with slight increase in ϕ , Nu_0 decreases while Nu_1 increases. Again, with increase in c , Nu_0 increases while Nu_1 decreases.

Table II: Numerical values for skin friction on the channel plates

λ	$\varepsilon = 0.01, R_T = 0.01,$ $\phi = 0.1, c = 0.1,$ $Gr = 10, M = 1$ τ_0 τ_1	$\varepsilon = 0.04, R_T = 0.01,$ $\phi = 0.1, c = 0.1,$ $Gr = 10, M = 1$ τ_0 τ_1	$\varepsilon = 0.04, R_T = 0.04,$ $\phi = 0.1, c = 0.1,$ $Gr = 10, M = 1$ τ_0 τ_1	$\varepsilon = 0.04, R_T = 0.04,$ $\phi = 0.1, c = 0.3,$ $Gr = 10, M = 1$ τ_0 τ_1
0.1	2.83322 2.63734	2.81556 2.62253	2.84510 2.66538	2.93564 2.44038
0.3	3.91679 2.73376	3.89354 2.71849	3.90334 2.75920	4.08705 2.53810
0.5	5.19877 2.86072	5.17240 2.84718	5.19630 2.87401	5.46283 2.76598
0.7	7.02756 3.27627	7.00799 3.27001	7.03197 3.26943	8.13084 3.80993
0.9	11.3904 5.53925	11.4668 5.59259	11.7015 5.65006	33.0152 22.7099

Table II reflects the effect of varying parameters on the skin frictions between the working fluid and the channel plates. It is viewed from the table that for some fixed values of parameters; the skin friction on the plates increases with increase in λ . Furthermore; with small increase in ε , c and R_T ; the skin friction on all the plates increases with increase in λ .

3.0 Validation of the result:

To authenticate the validity of the result; the present result on setting $\lambda = \varepsilon = c = R_T = 0$, $M = 1$ is compared with that of the published work of Jha and Ajibade [35] on relaxing heat absorbing/generating parameter and the comparison is tabulated below:

Table III:

Jha and Ajibade [35] when $S = 0$			Present study when $R_T = \varepsilon = 0$ and $M = 1$	
y	$\theta(y)$	$u(y)$	$\theta(y)$	$u(y)$
0.1	0.9000	1.1850	0.9000	1.1850
0.3	0.7000	1.3000	0.7000	1.3000
0.5	0.5000	1.1250	0.5000	1.1250
0.7	0.3000	0.7600	0.3000	0.7600
0.9	0.1000	0.2650	0.1000	0.2650

Numerical values in table III above show that the two studies have good agreement between them.

5.0 Conclusion:

Natural convection Couette flow through a vertical porous channel with thermal radiation and variable fluid properties effects was investigated using non-linear Rosseland heat diffusion, Adomian decomposition method and computer algebra package where results were presented and discussed. The major findings of the investigation are:

- i. The velocity and temperature of the fluid within the channel were found to decrease with increase in suction parameter.
- ii. Both the fluid velocity and temperature in the channel were discovered to descend with increase in thermal conduction parameter.
- iii. A decrease in the fluid viscosity was realized to ascend the fluid velocity within the channel.

- iv. Increase in thermal radiation parameter was recognized to rise both the fluid's temperature and velocity within the channel.

6.0 Nomenclature and Greek symbols:

<i>Symbols</i>	<i>Interpretation</i>	<i>Unit</i>
y'	Dimensional length	m
y	Dimensionless length	
g	Gravitational acceleration	ms^{-2}
k	Thermal conductivity	W/mK
T	Dimensional temperature	K
h	Dimensional channel width	m
T_w	Wall temperature	K
T_0	Ambient temperature	K
u, v	Dimensional velocity	ms^{-1}
ν	Kinematic viscosity	m^2s^{-1}
α	Thermal diffusivity	m^2s^{-1}
δ	Absorption coefficient	
β	Volumetric expansion coefficient	K^{-1}
μ	Dynamic Viscosity	$kgm^{-1}s^{-1}$
μ_0	Ambient fluid viscosity	$kgm^{-1}s^{-1}$
ε	Thermal conductivity variation parameter	
k_0	Ambient thermal conductivity	

q_r	Radiative heat flux	Wm^{-2}
ϕ	Temperature difference parameter	K
θ	Dimensionless temperature	
σ	Stefan-Boltzman constant	JK^{-1}
λ	Viscosity variation parameter	
\Re	Set of real numbers	
c	Suction parameter	
S	Heat generation/absorption parameter	
R	boundary condition at $y = 1$	
Nu_0	Nusselt number on the heated plate	
Nu_1	Nusselt number on the cold plate	
τ_0	Skin friction on the heated plate	
τ_1	Skin friction on the cold plate	
M	Velocity of moving boundary	

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