

# DESIGN OF MEDICAL WASTES INCINERATOR FOR HEALTH CARE FACILITIES IN AKURE

## ABSTRACT

Health Care Facilities (HCFs) are primarily saddled with the responsibilities of providing medical care, thus ensuring sound health of individuals. Tremendous efforts have been made by the government to ensure her availability in nooks and crannies of every community, which have resulted into improved medical services. However, among other environmental challenges confronting health care facilities in developing countries is Medical Waste generated in the course of carrying out their duties which is often ignored and in most instances treated as municipal or domestic solid waste. Effective management of medical waste requires keen planning, training and tracking throughout the waste generation, segregation, storage, collection, transportation, treatment and disposal processes. The fundamental information for selecting and designing the most efficient treatment method of medical waste is obtained by means of Waste Composition Analysis. Results from this study revealed that the daily waste generation rate of Ondo State Specialist Hospital Akure (OSSHA) and Mother and Child Hospital Akure (MCHA) was 124.5 kg/day. The hospitals' waste consists of 81.6% combustible wastes and 18.4% non-combustible wastes by mass. The combustible wastes are paper (6.50%), textiles (14.34%), cardboard (3.88%), plastics (6.04%) and food waste (19.08%). Since the ratio of combustible medical waste is higher than non-combustible medical waste, incineration (thermal destruction) at elevated temperature under controlled operational condition is considered the best disposal option to detoxify the medical waste. In other to prevent the release of harmful gases from burnt

25 medical waste through incinerator, a counter-current packed bed wet scrubber is designed  
26 which operates by impaction and absorption.

27 **Keywords:** Health Care Facilities, Medical waste, incinerator, waste composition analysis

## 28 **1.0 INTRODUCTION**

29 Health-care facilities generate medical waste which is capable of creating unsafe environment  
30 for both man and animals, as well as alter the properties of soil and local groundwater.  
31 Management of a medical waste thus becomes a matter of concern to public health  
32 administrator, environmentalist, infection control specialists, as well as the populace due to  
33 its potential environmental hazards and public health risks as it contains highly toxic  
34 chemicals, bacteria and pathogenic viruses [1, 2]. It is undoubted that health-care activities  
35 generate various types of hazardous and infectious materials. However, the consequences of  
36 indiscriminate disposal of medical waste have been highlighted by various regional and  
37 global studies, but the methods to manage this waste in a scientific manner putting into  
38 consideration safety of the ecosystem have not been fully introduced [3, 4, 5]. As a result,  
39 majority of the health institutions disposed combustible and non-combustible medical waste  
40 by open burning together with domestic waste, a practice considered inimical to the health of  
41 nearby dwellers [6, 7]. In Nigeria, biomedical wastes are characterized as infectious wastes  
42 which are further categorized as pathological waste, culture and stock of infectious agents,  
43 sharps (hypodermic needles, syringes and scalpel blades), waste from human blood, waste  
44 from surgery or autopsy that were in contact with infectious agents and products of blood and  
45 laboratory waste [8, 9]. Other wastes in these category includes waste from diverse  
46 therapeutic operations such as dialysis, autopsy, chemotherapy and biopsy para clinical test  
47 which generates chemical, radioactive and toxic materials that affect the environment and her  
48 occupants [2]. Every health-care facility is expected to effectively manage their waste  
49 following the right processes from the point of generation to final disposal [6]. Incineration of

50 Medical Waste has many benefits such as significant volume reduction (about 90%) and mass  
51 reduction (about 70%), thorough disinfection and energy recovery. Thus, incineration ensures  
52 detoxification, decrement and resource recovery, and it has been technically proven as an  
53 reliable waste treatment method [5, 10, 11].

#### 54 **1.1 Incineration Technology**

55 Disposal of medical waste through incineration process has been widely accepted in the field  
56 of infectious and hazardous waste management with regards to its advantages, which includes  
57 reduction in the quality (infectious state) and quantity (weight and volume) of the waste,  
58 reduction in toxic emission, suitability for all types of waste apart from sharps, exclusion of  
59 the risk of contamination of soil and local groundwater and low construction cost [2]. Waste  
60 obtained from hospitals is heterogeneous in nature because they consist of various degrees of  
61 elements in major and minor quantities, some of which are toxic and extremely infectious if  
62 not properly managed [4, 9, 12]. Hence, the need for incineration to decontaminate the  
63 medical waste by subjecting it to thermal destruction process at high temperature (1100°C -  
64 1600°C) under controlled operational conditions. The products of combustion are ash residue,  
65 water and carbon-dioxide. Incinerator is the unit in which the process occur. A well-designed  
66 incinerator does not only consider reduction of waste volume as priority but the environment  
67 as well must be put into consideration, hence, the need for incorporation of a gas cleaning  
68 device to the incineration process to ensure the release of clean and safe air to the  
69 atmosphere. A complete combustion of the medical waste and reduction in potential  
70 pollutants contained in the emission lends the process well to waste disposal in areas where  
71 population density is relatively high and availability of sites for landfill is low [13, 14].  
72 Incinerators reduce the solid mass of the initial waste by 80–85% and compresses the volume  
73 by 95–96%, based on the composition and extent of recovery of the material. Thus, as

74 incineration does not replace landfilling completely, it reduces the required volume for  
75 disposal definitely [15, 16, 17].

76 Minimization of the impacts of medical waste in HCFs is pre-requisitely a function of  
77 appropriate and practicable waste management system. Ethically, it is the responsibility of  
78 HCFs management to ensure proper medical waste management, which involves the  
79 determination of sources, waste characterization, frequency of generation, safe handling  
80 practices, segregation, storage, transportation, treatment and final disposal [1, 2, 15]. Most  
81 dominant approach to medical waste treatment and disposal in Africa, Asia and some parts in  
82 Europe are landfill, open burning and incineration. However, most of the HCFs often neglect  
83 the harmful side of these practices when it is not duly followed according to the World Health  
84 Organisation standard [3, 5, 8]. The use of incinerator without flue gas emission control  
85 device is as good as burning the waste in open space while unengineered landfill is  
86 synonymous to direct contamination of groundwater. Therefore, this study is geared towards  
87 design and the development of a medical waste incinerator equipped with a counter-current  
88 packed bed wet scrubber.

## 89 **2.0 MATERIAL AND METHODS**

90 The medical waste incinerating system equipped with air pollution control device for the  
91 health care facilities in Akure, Ondo state, Nigeria was designed and developed using  
92 appropriate and essential principles. The fabrication of the system is in progress. The major  
93 component of the incineration system includes the combustion system, connecting ducts,  
94 filtration system and the air pollution control system.

### 95 **2.1 Design of a Controlled-Air-Batch-Feed Incineration Technology**

#### 96 **2.1.1 Determination of the Incinerator Capacity**

97 The incinerator capacity and burning time (residence time) was determined from the quantity  
98 of waste load generated by the HCFs using the equations developed by Walter [18]:

$$Y = 1.72 \times W^{0.76} \quad (1.)$$

$$N = \frac{W}{0.9Y} \quad (2.)$$

99 Where W is the waste load (lbs/day or kg/day), Y is the optimum incinerator capacity (lbs/hr  
 100 or kg/hr) and N is the Optimum burning time (hrs/day). From the survey and measurement,  
 101 the average wastes quantification from the two public hospitals is shown in Table 1 below.

102 Table 1: Quantity of medical wastes generated daily and monthly in the HCFs

S/N	Type of health care facility	Quantity of waste generation (Kg/day)	Quantity of waste generation (Kg/month)
1	Ondo State Specialist Hospital, Akure	81.2	2,436
2	Mother and Child Hospital, Akure	43.3	1,299
	Total Wasste Generated	124.5	3,735

103 For unknown future of higher generation of medical waste, the waste load (W) from the  
 104 HCFs is estimated as 269 kg/day. Hence the optimum incinerator capacity (Y) is 100 kg/hr.

### 105 2.1.2 Design assumptions

106 For the air-starved batch-feed type of incinerator designed to treat a mixture of 70% ‘black  
 107 bag’ and 30% ‘red bag’ medical wastes at a optimum throughput capacity of 100 kg/hr, the  
 108 following assumptions were made with regards to United States Environmental Protection  
 109 Agency [19].

- 110 ▪ Ignition/Primary chamber temperature is 760°C (1400°F)
- 111 ▪ Secondary chamber temperature is 1100°C (2010°F)
- 112 ▪ Flue gas residence time at 1000°C (1830°F) is 1 second
- 113 ▪ Residual oxygen in flue gas is 6% minimum
- 114 ▪ 30% of air required for stoichiometric combustion is supplied into the primary chamber

- 115     ▪ Excess air at 150% of the theoretically required air is supplied in the secondary chamber  
116         of the incinerator during the peak burning rate.
- 117     ▪ Detailed monitoring of the temperatures of gases and water and at critical points of the  
118         system with the use of appropriate devices.
- 119     ▪ The use of thick standard materials for adequate protection of the combustion chamber  
120         from fire.
- 121     ▪ A little opening (sealed glass covered by blast-gates) as view point is installed on the  
122         primary chamber to enhance easy view of the flame pattern and the waste bed.
- 123     ▪ The use of adequate refractory and insulation materials for combustion chambers outside  
124         surfaces to maintain operating temperature.
- 125     ▪ The burners are rightly positioned in the primary chamber to provides maximum  
126         impingement of the flame onto the wastes achieving a minimum supply of 80% of the  
127         total heat input of the incinerator design capacity.

### 128     **2.1.3 Design of primary chamber**

129     In design of the primary chamber of the starved-air-batch-feed incinerator, the initial volume  
130     of the chamber is determined. The optimum incinerator capacity per hour (100kg/hr) was  
131     dumped as a heap and the volume is calculated as slightly rounded parabolic shape measured  
132     as 5 m<sup>3</sup> value of which is used in the design of the chamber using equation [20].

$$V = L \times B \times H \qquad (3.)$$

133     Assuming a suitable chamber depth, H of 2 m, with the ratio of length to breadth as 1.5:1,

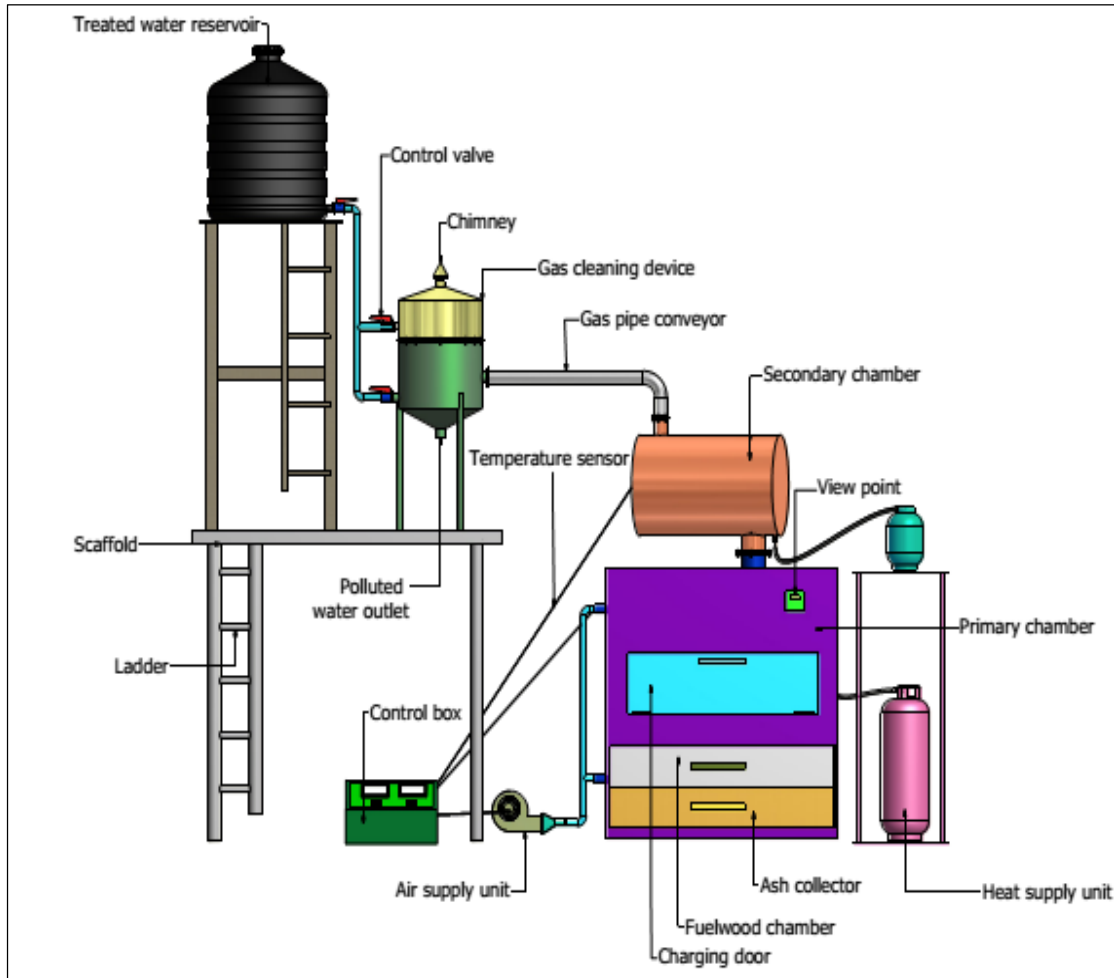
134     Hence, the width of incinerator, B is 1.29 m and the length of incinerator, L is 1.93 m.

135     Chamber sizing is based on heat release, which is the amount of heat generated when  
136     combustible material burns. [7, 17]. Biomedical waste contains varieties of low density, high  
137     heating value wastes (e.g. plastics), as well as high density, low heating value wastes (e.g.  
138     tissue, bones). Therefore, the primary chamber was sized to accommodate the variation in the

139 waste composition. The volume of the primary chamber was designed to allow for a total heat  
140 release rate of 41,020.73 kJ/h and with an operating temperature range between 400-760°C  
141 (750-1400°F).

## 142 **2.2 Machine Conception**

143 The main concept behind this machine is to design a movable and well-regulated incineration  
144 system equipped with air pollution control device that will be economically feasible and  
145 environmental friendly. The machine is expected to be used basically for burning solid  
146 combustible medical wastes at 400°C – 760°C and 1100°C in the primary and secondary  
147 combustion chambers respectively. The material selected for the designed are locally  
148 available which makes the cost of production low. The incinerator uses basic principle of  
149 conduction to achieve burning while flue gas emission control device utilises the principle of  
150 absorption and impaction with the aid of counter-current randomly-packed bed and water  
151 (wet scrubber) to remove hazardous/infectious substances from emitted gases. It is expected  
152 that the machine reduces the quantity and quality of the medical waste after burning to  
153 produce ash and harmless gases. A conceptual drawing of the machine is shown in figure 1.



154

155 **Figure 1: Conceptual drawing of a batch feed incinerator incorporated with packed bed**  
 156 **wet scrubber**

157 **2.3 Heat and Material Balance**

158 Heat and material balance calculation is an integral part of designing and evaluating  
 159 incinerators. The technique involves a detailed estimation of the input and output conditions  
 160 of the incinerator. It was used to determine the combustion air and auxiliary fuel  
 161 requirements for incinerating a given medical waste and/or to determine the limitations of an  
 162 existing incinerator when charged with a known waste [4, 20]. The following steps were  
 163 taken to calculate the heat and material balance sample.

164 **2.3.1 Heating Values of Material Input**



165 The material flow per hr into the incinerator is 100kg/hr. Based on an input of 30% of 100  
 166 kg/h (i.e 30 kg/h), the ‘red bag’ is assumed to have the following composition, according to  
 167 Oumarou *et al.* [17] and John and Swamy [20]: dry tissue, water and ash represents 6.0, 21.0  
 168 and 3.0 kg/h respectively.

169 The black bag waste input is 70% of 100.02 kg/h (i.e 70 kg/h) is assumed to consist of  
 170 polyethylene, polyvinylchloride, cellulose, ash in the proportion of 21.0 kg/h, 2.1 kg/h, 36.4  
 171 kg/h, 10.5 kg/h respectively as shown in Table 2.

172 **Table 2: Higher heating values and total heat of the combustible medical waste**

Component	Calorific value kcal/g	HHV kJ/kg	Input kg/h	Total Heat in kJ/h
$C_5H_{10}O_3$	6.028	25,220	6.0	151,320.0
$H_2O$	0.0	0.0	21.0	0.0
$(C_2H_4)_x$	9.039	37,820	21.0	794,220.0
$(C_2H_3Cl)_x$	9.119	38,154	2.1	80,123.4
$C_6H_{10}O_5$	5.703	23,860	36.4	868,504.0
Ash	0.0	0.0	13.5	0.0
<b>Total</b>			<b>100.0</b>	<b>1,894,167.4</b>

173 **2.3.2 Determination of Stoichiometric Oxygen for combustible medical Wastes and**  
 174 **Combustion air rates**

175 The total stoichiometric (theoretical) amount of oxygen required to oxidize (burn) the waste  
 176 is determined by the chemical equilibrium equations of the individual components of the  
 177 biomedical waste from laboratory analysis, the stoichiometric oxygen required to burn the  
 178 combustible component of the biomedical waste (100 kg/h) is shown in Table 3.

179 **Table 3: The combustion equation and the stoichiometric air requirement**

Waste	Combustion Equation	Stoichiometric air requirement (per kg waste) (kg/hr)
Tissue (dry)	$C_5H_{10}O_3 + 6O_2 \Rightarrow 5CO_2 + 5H_2O$	9.34
Poly Ethylene	$(C_2H_4)_x + 3O_2 \Rightarrow 2CO_2 + 2H_2O$	79.5
PVC	$2(C_2H_3Cl)_x + 5O_2 \Rightarrow 4CO_2 + 2H_2O + 2HCl$	4.27
Cellulose	$C_6H_{10}O_5 + 6O_2 \Rightarrow 6CO_2 + 5H_2O$	41.6
		134.8

180 The stoichiometric air is calculated thus:

$$\text{Stoichiometric air} = \text{Stoichiometric } O_2 \times \frac{\text{Total Input (kg/h)}}{\text{Molecular weight of } O_2} \quad (4.)$$

$$= 134.8 \times \frac{100}{23} = 586.1 \text{ kg/h of air}$$

181 Primary chamber was supplied at 30 % of that required for stoichiometric combustion total  
182 air required for waste at primary chamber =  $(0.3 \times 586.1) + 586.1 = 761.9 \text{ kg/h}$ . The air  
183 supply in the secondary chamber was designed to provide excess air 150 % of that  
184 theoretically required during the peak burning rate [3, 11, 20]. Hence, Total air required for  
185 waste at secondary chamber (150% excess) =  $(1.5 \times 586.1) + 586.1 = 1465.3 \text{ kg/h}$ .

### 186 2.3.3 Material Balance for Combustion Chambers.

#### 187 A. Total Mass Input

188 Total air required for waste at secondary chamber (Dry air) = 1465.3 kg/h

189 Total mass of waste per hour = 100 kg/h

190 Moisture in air = mass of oxygen in air x dry air (5.)

$$191 = 0.0132 \times 1465.3 = 19.3 \text{ kg/h}$$

192 Total Mass input = mass of waste + Dry air + moisture in air (6.)

$$193 = 100 \text{ kg/h} + 1465.3 \text{ kg/h} + 19.3 \text{ kg/h} = \mathbf{1584.6 \text{ kg/h}}$$

#### 194 B. Total Mass Output (Assuming Complete Combustion)

195 ***Dry Products from waste***

196 Less stoichiometric air for waste = 586.1kg/h

197 Total excess air = 586.1 x 1.5 = 879.2 kg/h

198 Adding nitrogen from stoichiometric air 0.77 x 586.1= 451.3 kg/h

199 Sub-total air = (879.2 + 451.3) = 1330.5 kg/h

200 Add total CO<sub>2</sub> from combustion:

201 CO<sub>2</sub> formed from C<sub>5</sub>H<sub>10</sub>O<sub>3</sub> = 10.47 kg/h

202 CO<sub>2</sub> formed from (C<sub>2</sub>H<sub>4</sub>)x = 72.4 kg/h

203 CO<sub>2</sub> formed from (C<sub>2</sub>H<sub>3</sub>Cl)x = 3.92 kg/h

204 CO<sub>2</sub> formed from C<sub>6</sub>H<sub>10</sub>O<sub>5</sub> = 56.2 kg/h

205 142.9 kg/h

206 Total waste dry products = Sub-total air + total CO<sub>2</sub> from combustion (7.)

207 = (1330.5 + 142.9) kg/h = 1473.4 kg/h

208 ***Moisture present in waste***

209 H<sub>2</sub>O in the waste = 21.0 kg/h

210 H<sub>2</sub>O from combustion reactions = 55.5 kg/h

211 H<sub>2</sub>O in combustion air = 19.3 kg/h

212 Total Moisture = 95.8 kg/h

213 ***Ash Output***

214 Ash Output = 13.5 kg/h

215 ***HCl formed from Wastes***

216 HCl formed from (C<sub>2</sub>H<sub>3</sub>Cl)x = 1.65 kg/h

217 ***Total Mass Out***

218 Total Mass Out = Total waste dry products + Total moisture + Total CO<sub>2</sub> from combustion +

219 HCl formed from waste + Ash Output (8.)

220 
$$= (1330.5 + 95.8 + 142.9 + 1.65 + 13.5) \text{ kg/h} = \mathbf{1584.4 \text{ kg/h}}$$

221 **2.3.4 Energy balance of the incinerator**

222 Analysis of energy balance for an incinerator prototype entails the use of first law of  
 223 thermodynamics and energy conservation [21] i.e.;

$$\Sigma E_{\text{input}} = \Sigma E_{\text{output}} \quad (9.)$$

224 **A. Total energy input to the incinerator**

225 
$$\text{Energy}_{\text{input}} = Q_{\text{bmw}} + Q_{\text{Natural gas}} + Q_{\text{air}} \quad (10.)$$

226 Assuming energy from air  $Q_{\text{(air)}}$  is negligible we have the energy input to be:

227 
$$\text{Energy}_{\text{input}} = Q_{\text{bmw}} + Q_{\text{Natural gas}} \quad (11.)$$

228 Total heat in from combustible medical waste  $Q_{\text{bmw}}$  is the summation of all combustible  
 229 materials:

230 
$$\text{Total heat required to burn cellulose} = 868,504 \text{ kJ/h}$$

231 
$$\text{Total heat required to burn Rubber} = 794,220 \text{ kJ/h}$$

232 
$$\text{Total heat required to burn Plastic} = 80,123.4 \text{ kJ/h}$$

233 
$$\text{Total heat required to burn Tissue} = 151,320 \text{ kJ/h}$$

234 
$$Q_{\text{bmw}} = \text{Total heat required input} = 1,894,167.4 \text{ kJ/h}$$

235 Total heat in from natural gas  $Q_{\text{Natural gas}}$  is calculated as:

236 
$$Q_{\text{Natural gas}} = \text{Energy from natural gas}$$

237 
$$\text{Mass flow rate of natural gas} = 20.6 \text{ kg/h (assumed)}$$

238 
$$= \text{mass flow rate of natural gas} \times \text{higher heating value of natural gas}$$

239 
$$= 20.6 \text{ kg/h} \times 43,000 \text{ kJ/kg} = 885,800 \text{ kJ/h}$$

240 Hence, the total energy supplied to the system = (1,894,167.4 + 885,800) kJ/h

$$\text{Energy}_{\text{in}} = 2,779,967.4 \text{ kJ/h}$$

241 **B. Total energy output from the incinerator**

242 
$$\text{Energy}_{\text{output}} = Q_{\text{et}} + H_{\text{flue gases}} \quad (12.)$$

243 Total heat out based on equilibrium temperature of 1100°C ( $Q_{et}$ )  
 244 Radiation loss ( $R_{loss}$ ) = 5% of total heat available (13.)  
 245  $= 0.05 \times 1,894,167 \text{ kJ/h} = 94,708.35 \text{ kJ/h}$

246 Heat to ash, Heat to dry combustion products and Heat out due to flue gases release is  
 247 calculated using equation 14 as used by Patel and Kumar [4], Ganguly *et al.* [12] and Walter  
 248 [18];

$$\Delta H = mC_p\Delta T \quad (14.)$$

249 Heat to ash ( $H_{ash}$ ) is calculated as 12,166.5 kJ/h using equation 14, Where weight of ash,  $m =$   
 250 13.5 kg/h, mean heat capacity of ash,  $C_p = 0.831 \text{ kJ/kg}^\circ\text{C}$  ([20] and Temperature difference,  
 251  $\Delta T = 1084.5^\circ\text{C}$ .

252 Heat to dry combustion products ( $H_{dcp}$ ) is then calculated as 1,735,321.9 kJ/h using equation  
 253 14, where weight of combustion products,  $m = 1473.4 \text{ kg/h}$ , mean heat capacity of dry  
 254 (medical wastes) products,  $C_p = 1.086 \text{ kJ/kg}^\circ\text{C}$  and Temperature difference,  $\Delta T = 1084.5^\circ\text{C}$ .

255 Heat to moisture ( $H_{moisture}$ ) is then calculated as 479,538.5 KJ/h using equation 15,

$$\Delta H_{moisture} = mC_p\Delta T + mH_v \quad (15.)$$

256 Where weight of water,  $m_3 = 95.80 \text{ kg/h}$ , mean heat capacity of water,  $C_{p3} = 2.347 \text{ kJ/kg}^\circ\text{C}$ ,  
 257 Temperature difference,  $\Delta T = 1084.5^\circ\text{C}$ ,  $H_v =$  latent heat of vaporizations of water = 2460.3  
 258 kJ/kg.

259 Heat out due to flue gases release ( $H_{flue \text{ gases}}$ ) is then calculated as 266,570.7 kJ/h using  
 260 equation 4, which involves the addition of the Heat out due to release of  $\text{CO}_2$ ,  $\text{O}_2$ ,  $\text{HCl}$   
 261 respectively, where the mass,  $m$  are 142.9 kg/h, 134.8 kg/h and 1.65 kg/h; specific heat

262 capacity,  $C_p$  are 0.844 kJ/kg°C, 0.919 kJ/kg°C and 0.795 kJ/kg°C for CO<sub>2</sub>, O<sub>2</sub> and HCl  
263 respectively; and Temperature difference,  $\Delta T$  is 1084.5°C.

264 Hence, the total Energy<sub>out</sub> ( $Q_{out}$ ) = ( $R_{loss} + H_{ash} + H_{dcp} + H_{moisture} + H_{flue\ gases}$ ) kJ/h (16.)

$$\text{Energy}_{out} = 2,588,305.9 \text{ kJ/h}$$

### 265 **2.3.5 Determination of auxiliary fuel required to achieve 1100°C and mass flow rate**

266 Total heat required from natural gas ( $H_{fuel}$ ) = 2,779,967.4 + 5% radiation loss = 2,918.965.8

267 kJ/h, Available heat (net) from natural gas at 1100°C and 20% excess air = 89,814.3 kJ/m<sup>3</sup>

268 (Assumed). Therefore, natural gas required is (2,918.965.8 kJ/hr / 98,947.99 kJ/m<sup>3</sup>) = 29.5

269 m<sup>3</sup>/h [14, 20].

270 Mass flow rate of gas,  $\dot{m}$  = density of methane gas,  $\rho_a$  x volumetric flow rate of gas

$$271 = 1.25 \text{ kg/m}^3 \times 29.5 \text{ m}^3/\text{h} = 36.88 \text{ kg/h} = 0.01 \text{ kg/s.}$$

272 Assuming 10 gas burners, each burner will consume 3.69 kg of methane gas per hour which

273 is equivalent to 0.0011 kg/s. Six burners in primary chamber will deliver at 0.0066 kg/s

274 (22.14 kg/h) and four burners in secondary chamber at 0.0044 kg/s = 14.74 kg/h. Dry

275 Products from fuel at 20% excess air = dry fuel density x Methane gas required = 16.0 kg/m<sup>3</sup>

276 x 29.5 m<sup>3</sup>/hr. fuel = 472 kg/h [14] [20].

### 277 **2.3.6 Secondary chamber volume required to achieve 1 second residence time at 1100**

278 °C

279 Total dry product which is the summation of the Total Dry Products from waste and Dry

280 Products from fuel = (1,584.4 + 472) kg/h = 2056.4 kg/h, assuming dry products have the

281 properties of air and using the ideal gas law, the volumetric flow rate ( $V_p$ ) of dry products

282 ( $d_p$ ) at 1000°C is calculated as 2.1 m<sup>3</sup>/s using equation 17 [22].

$$283 PV_p = nR_p T_p \quad (17.)$$

284 ii) Total moisture = Total moisture from waste + Moisture from fuel

$$285 = 95.8 \text{ kg/h} + 46.9 \text{ kg/h} = 142.7 \text{ kg/h}$$

286 Using the ideal gas law, the volumetric flow rate of moisture at 1000°C ( $V_m$ ) as 0.2 m<sup>3</sup>/s  
287 using equation 17. Total volumetric flow rate ( $V_t$ ) which is the summation of volumetric flow  
288 rate of dry products at 1000°C and volumetric flow rate of moisture at 1000°C is 2.3 m<sup>3</sup>/s.  
289 Therefore, the active chamber volume required to achieve one second retention is 2.3m<sup>3</sup>.

290 The observed one second retention time of 2.3m<sup>3</sup> is sufficient for the active chamber.  
291 Although, some dead spaces occur in the chamber creating zero or negligible flow in reality.  
292 Hence, the length of the secondary chamber is calculated from the flame front to the location  
293 of the temperature sensor to achieve the retention time of one second as recommended by  
294 Ganguly *et al.* [12] and Walter [18]

## 295 **2.4 Residual Oxygen in the Flue Gas**

296 The residual oxygen (% O<sub>2</sub>) was determined by taking 21% of mass flow rate of air used  
297 through the following equation:

$$\text{EA (excess air)} = \frac{\%O_2}{(21\% - \%O_2)} \quad (18.)$$

$$\frac{150}{100} = \frac{\%O_2}{(21\% - \%O_2)}$$

298  $150 (21\% - \%O_2) = 100 \%O_2$

299  $\%O_2 = 12.6\%$

## 300 **2.5 Efficiency of the machine**

301 The efficiency of the machine is calculated using the relation:

$$\eta_{\text{incinerator}} = \frac{\text{Energy output}}{\text{Energy input}} \times 100\%$$

$$= \frac{2,588,305.9}{2,779,967.4} \times 100 = 93\%$$

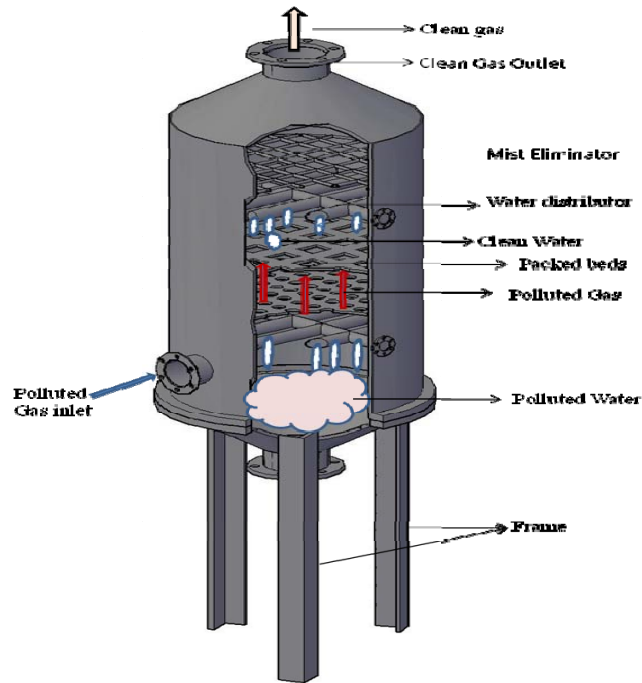
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## 303 **2.6 Air Pollution Control System**

### 304 **2.6.1 Design conception of packed bed wet Scrubber**

305 A vertical design concept is considered for the packed beds wet scrubber, the liquor is  
306 sprayed from the top and flows downward across the bed. Appropriate distribution of liquor  
307 is important for efficient removal of gases [22]. The collection of acid gases in packed-bed  
308 scrubbers is achieved by absorption. The effectiveness of absorption in packed beds is related  
309 to the uniformity of the gas velocity distribution, the surface area of the packing material, the  
310 amount and uniform distribution of scrubber liquid, and the pH and turbidity of the scrubbing  
311 liquid. The measure of gas absorption is affected by the extensive liquid surface contacted by  
312 the gas stream as the liquid flows downward over the packing material [16, 18]. Variety of  
313 available packing materials offer a large exposed surface area to facilitate contact with and  
314 absorption these acid gases. The packing materials which ranges in size from 0.5 to 3 inches  
315 are randomly oriented in the bed. Typically, sodium hydroxide (NaOH) or occasionally  
316 sodium carbonate ( $\text{Na}_2\text{CO}_3$ ) is used with water to neutralize the absorbed acid gases in a  
317 packed-bed scrubber. These two soluble alkali materials are preferred because they minimize  
318 the possibility of scale formation in the nozzles, pump, and piping. For the typical case of  
319 using NaOH as the neutralizing agent, the HCl and  $\text{SO}_2$  collected in the scrubber react with  
320 NaOH to produce sodium chloride (NaCl) and sodium sulphite ( $\text{Na}_2\text{SO}_3$ ) in an aqueous  
321 solution. The conceptual design of the countercurrent packed bed wet scrubber is shown in  
322 figure 2 below.





323

324 Figure 2: Conceptual drawing of a countercurrent packed bed wet scrubber

### 325 2.6.2 Design Analysis of Packed-Bed Scrubber/Absorber

326 The following assumptions were made in the design of packed tower scrubber:

327 ▪ Pollutant concentration entering the column in the waste gas  $Y_1 = 0.07$

328 ( $k - mol\ pollutant\ gas / k - mol\ pollutant\ waste\ gas$ )

329 ▪ Pollutant concentration entering the column in the solvent (liquid phase)  $X_1 = 0.000005$

330 ( $k - mol\ pollutant\ gas / k - mol\ pollutant\ solvent$ )

331 ▪ Maximum concentration of the pollutant in the liquid phase if it were allowed

332 to come to equilibrium with the pollutant entering the column in the gas phase  $X_1^* = 0.55$

333 ( $k - mol\ pollutant\ gas / k - mol\ pollutant\ waste\ gas$ )

334 ▪ Waste gas flow rate entering the column  $G_i = 1.259\ m^3/min$ .

335 ▪ Liquid flow rate entering the column  $L_i = 0.078\ m^3/min$

336 ▪ Efficiency of the scrubber  $\eta = 99\%$

337 Pollutant exiting the column in the waste gas can be determined from assumed efficiency of

338 air pollution control device using equation 19 as recommended by Walter [18] and Danzomo

339 *et al.* [23].

$$Y_O = Y_i(1 - \eta/100) \quad (19.)$$

340  $Y_O = 0.0007 \text{ mol}$

341 Minimum liquid to gas phase ratio is calculated as 0.126 using equation 20 as used by

$$\left[\frac{L_S}{G_S}\right]_{\min} = \left[\frac{Y_i - Y_O}{X_O^* - X_i}\right] \quad (20.)$$

342 Therefore, the actual liquid to gas phase ratio, which is a product of the minimum liquid to  
343 gas ratio and an adjustment factor,  $Adj_{\text{fac}}$  (usually between 1.2 and 1.5) is calculated as  
344 0.1512 using equation 21.

$$\left[\frac{L_S}{G_S}\right]_{\text{act}} = \left[\frac{L_S}{G_S}\right]_{\min} \times Adj_{\text{fac}} \quad (21.)$$

345 The waste flow rate of the gas,  $G''$  through the scrubber is determined as 0.0233 kg/s using  
346 equation 22, where  $\rho_g$  = density of combustion gases at STP = 1.11 kg/m<sup>3</sup> [22]

$$G'' = \frac{\rho_g \times G_i}{60} \quad (22.)$$

347 Hence, the gas velocity,  $V_{GF}$ , which is determined by dividing the waste gas flow rate by the  
348 density of the gas, is calculated as 0.0209 m/s.

349 The liquid flow rate,  $L''$  through the scrubber is determined as 1.3 kg/s using equation 23,  
350 where  $\rho_L$  = density of liquid at STP = 1000 kg/m<sup>3</sup> [20, 22].

$$L'' = \frac{\rho_g \times L_i}{60} \quad (23.)$$

351 The molar flowrate of the pollutant free solvent,  $L_S$  is estimated as 0.000114 kmol/s using  
352 equation 24, where the Molecular weight of gas,  $M_{wg}$  is 0.029 kg/mol [11, 18]

$$L_S = \left[\frac{L_S}{G_S}\right]_{\text{act}} \times \frac{G''}{M_{wg}(1 + Y_i)} \quad (24.)$$

### 353 2.6.3 Assumption of Absorption Factor

354 The Absorption factor ( $Abs_{fac}$ ) value is frequently used in describing the relationship between  
 355 the equilibrium line and the liquid-to-gas ratio [19]. In several pollutant-solvent systems, the  
 356 most economical value for  $Abs_{fac}$  ranges from 1.5 to 2.0 [16, 18]. For this design, the  
 357 adsorption factor of 1.65 is assumed.

#### 358 **2.6.4 Determination of superficial gas flow rate**

359 A Generalised pressure drop correlation chart showing the correlation between the liquid and  
 360 vapour flow rates, system physical properties and packing characteristics, with the gas mass  
 361 flow-rate per unit cross-sectional area; with lines of constant pressure drop as a parameter as  
 362 d used by Coker *et al.* [9]; Doherty and Malone [24], The Abscissa value from the graph is  
 363 calculated using Equation 25 as 0.06.

$$F_{LV} = \frac{L''}{G''} \sqrt{\frac{\rho_G}{\rho_L}} \quad (25.)$$

364 A percentage flooding of 80% is observed, which is satisfactory assuming a pressure drop of  
 365 8 mm H<sub>2</sub>O/m packing at Ordinate K<sub>4</sub> which is 0.4. The superficial gas flow rate/gas mass  
 366 flow-rate per unit column cross-sectional area,  $G_{sfr}$  is then determined using Equation 26 as  
 367 adopted by John and Swamy [20]; Doherty and Malone [24], where density of combustion  
 368 gases at STP,  $\rho_g$  is 1.11 kg/m<sup>3</sup> [22], density of scrubbing,  $\rho_L$  and acceleration due to  
 369 gravitational,  $g$  is 9.8 m/s (Gupta and [21]).

$$K_4 = \frac{13.1 (G_{sfr})^2 \times F_p \left[ \frac{\mu_L}{\rho_L} \right]^{0.1}}{\rho_g [\rho_L - \rho_g]} \quad (26.)$$

370 The superficial gas flow rate/gas mass flow-rate per unit column cross-sectional area,  $G_{sfr}$  is  
 371 then calculated as 0.5859 kg/m<sup>2</sup>s.

#### 372 **2.6.5 Determination of packed-bed area and diameter**

373 The value of the packed bed area and diameter on the actual gas flow rate per unit area is  
 374 estimated by determining the required column area,  $S_A$ . The required column area is

375 calculated by dividing the gas flow rate by the superficial gas flow rate as shown in equation  
376 27 below where Gas flow rate,  $G'' = 0.0233$  kg/s, mass flux of gas per cross sectional area of  
377 column,  $G_{sfr} = 0.5265$  kg/m<sup>2</sup>s. The column area required,  $S_A$  is 0.039 m<sup>2</sup> while the column  
378 diameter,  $D$  calculated as 0.223m using equation 28 as adopted by John and Swamy [20], and  
379 Danzomo *et al.* [23].

$$SA = \frac{\text{Gas flow rate } (G'')}{\text{superficial gas mass flow rate } (G_{sfr})} \quad (27.)$$

$$D = 1.13\sqrt{\text{Tower area}} \quad (28.)$$

380 The height of packing,  $Z$  is calculated by multiplying the assumed column height, 4m and the  
381 height of overall gas phase transfer unit,  $H_{OG}$  of 0.25m.

### 382 **3. CONCLUSION AND RECOMMENDATION**

#### 383 **3.1 Conclusion**

384 The estimated quantity of medical waste from surveyed health-care facilities was about 62.25  
385 kg/day, equivalent to 22.41 ton/year. The average generation rates of total medical waste,  
386 general waste, hazardous–infectious waste and sharp waste in public hospitals within Akure  
387 metropolis were 0.75, 0.50, 0.19 and 0.06 kg/bed/day respectively. Of the total medical waste  
388 generated in the facilities, 65.36% consisted of general waste, 26.78% was infectious waste,  
389 and 7.86% was sharps waste. The medical waste has higher calorific value, higher heating  
390 value and volatile matter, which can realize the sustained combustion of waste. The  
391 combustible component accounted for more than 60%, so it is entirely feasible to dispose  
392 medical waste by high temperature incineration.

393 Daily increment of medical waste generation and the quest to safeguard the people and  
394 environment from outbreak of diseases, a cost effective and environmental-friendly  
395 incinerator was designed in present study to treat biomedical waste generated in surveyed  
396 HCFs with a capacity of 100.0 kg/h. From the material balance analysis by assuming  
397 complete combustion, total mass input (1584.6 kg/h) is found to be equal to total mass output

398 (1584.4 kg/h) while the total energy input from the heat balance analysis is found to be  
399 2,779,967.4 kJ/h and total energy output to be 2,588,305.9 kJ/h. From the design analysis,  
400 184.7 m<sup>3</sup>/h of natural gas is required to achieve a design temperature of 1100°C. Also, from  
401 the design, the volume of secondary chamber is found to be 3.1 m<sup>3</sup> with a detention time of 1  
402 second. A Counter-current packed bed wet scrubber with 99% scrubbing efficiency was  
403 designed with the incinerator to adsorb toxic (flue) gases that might be emitted in the course  
404 of burning the waste.

### 405 **3.2 Recommendations**

406 This pilot study in Akure, the state capital territory of Ondo State, Nigeria, shows that very  
407 little has been done on medical waste management in the metropolis. It is therefore  
408 recommended that the Ministries of Environment and Health put in place a legislation that  
409 will regulate medical waste generation and management, and also adopt a multidisciplinary  
410 approach to manage medical waste generated within the metropolis and the state at large.  
411 Moreover, to improve the existing conditions, extensive research on effective waste  
412 management practices and regulation is paramount. However, the unending increment of  
413 medical waste generation due to multifaceted activities carried out in hospitals poses  
414 enormous environmental and health problems. to pose both environmental and health  
415 problems to the residents of the city, it is recommended that an energy recovery incinerator  
416 equipped with an air pollution control system is developed, positioned and used in treatment  
417 of wastes generated in the health care facilities in Akure, Ondo state, Nigeria.

418

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